

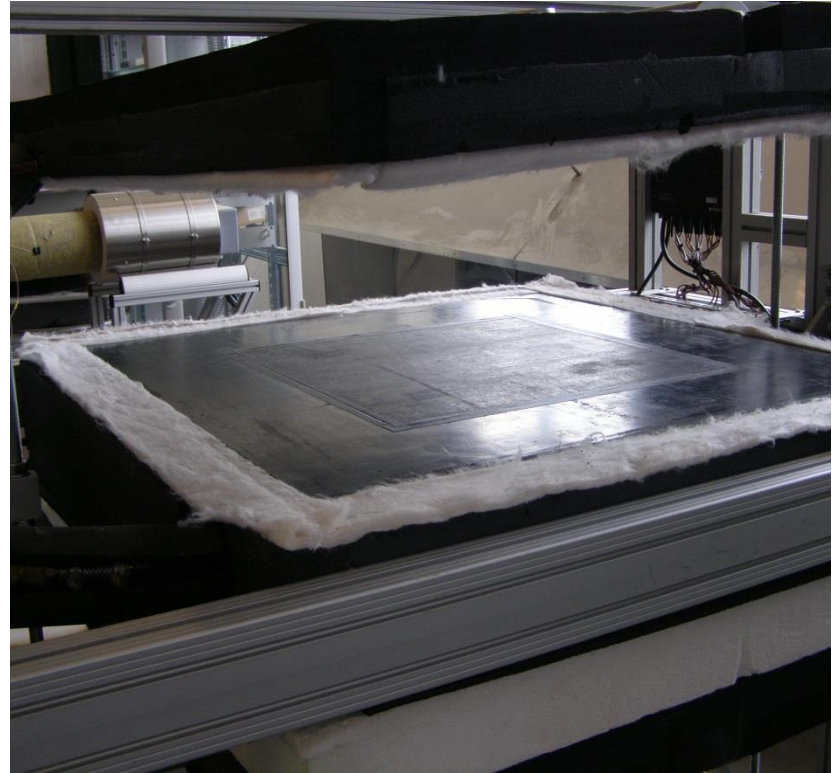
# Determination of linear thermal Transmittance of Vacuum-Insulation Panels by Measurement in a Guarded-Hot-Plate- (GHP) or a Heat-Flow-Meter- (HFM) Apparatus

Christoph Sprengard, FIW München



# Content

- Brief introduction on linear thermal transmittance  $\psi$ 
  - Building kit: how to get from  $\lambda_{\text{COP}}$  to  $\lambda_{\text{eff}}$  to R to U
  - Vice Versa: measure U or R and derive  $\lambda_{\text{eff}}$
- Measurement procedure for linear thermal transmittance  $\psi$
- Supporting FD-Simulations
- Results and Discussion



*Rotatable HFM apparatus of FIW*



# From $\lambda_{COP}$ to U...

...and from U back to  $\lambda_{eff}$  for declaration

# 2-dim and 3-dim thermal bridges

$$U_{eff} = \frac{Q_{eff}}{A_{panel} \cdot \Delta\vartheta}$$

$$\begin{aligned} Q_{eff} \\ = \Delta\vartheta \cdot (U_0 \cdot A_{panel} + \sum \psi_i \cdot l_i \\ + \sum \chi_i \cdot n_i) \end{aligned}$$

- Measure  $\lambda_{COP}$  in GHP or HFM apparatus and calculate  $U_0$
- Add 2-dim effects for edges of panels ( $\psi$  and length)
- 3-dim effects need to be taken into account in constructions ( $\chi$  and  $n$ )
- Effect easily exceeds the 3 % criterium in ISO 6946
- Calculate total heat flow
- Calculate  $U_{eff}$  depending on panel size and  $\Delta\vartheta$

# Derive $\lambda_{eff}$ from $U_{eff}$

$$\lambda_{eff} = \frac{d_{panel}}{\frac{1}{U_{eff}} - R_s}$$

$$\lambda_{eff} = \frac{d_{panel}}{R_{eff}}$$

- Derive  $\lambda_{eff}$  from
  - $U_{eff}$  from 2-/3-dim sim.+cal.
  - $U_{eff}$  from Hot-Box measurement
  - $R_{eff}$  from GHP or HFM measurement
  
- When calculating: all effects can be determined separately
  
- When using Hot-Box method: 2- and 3-dim effects can be considered (in sum!)
  
- When using GHP or HFM: only 2-dim effects can be measured (in sum!)



# Influencing Factors on $\psi$

## and order of magnitude

# Influencing Factors

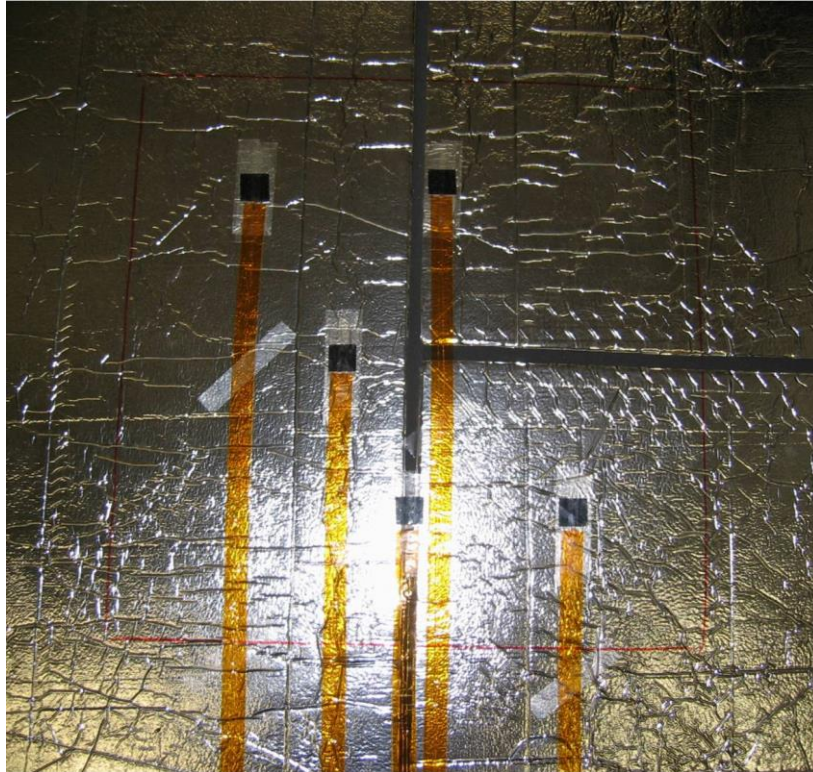
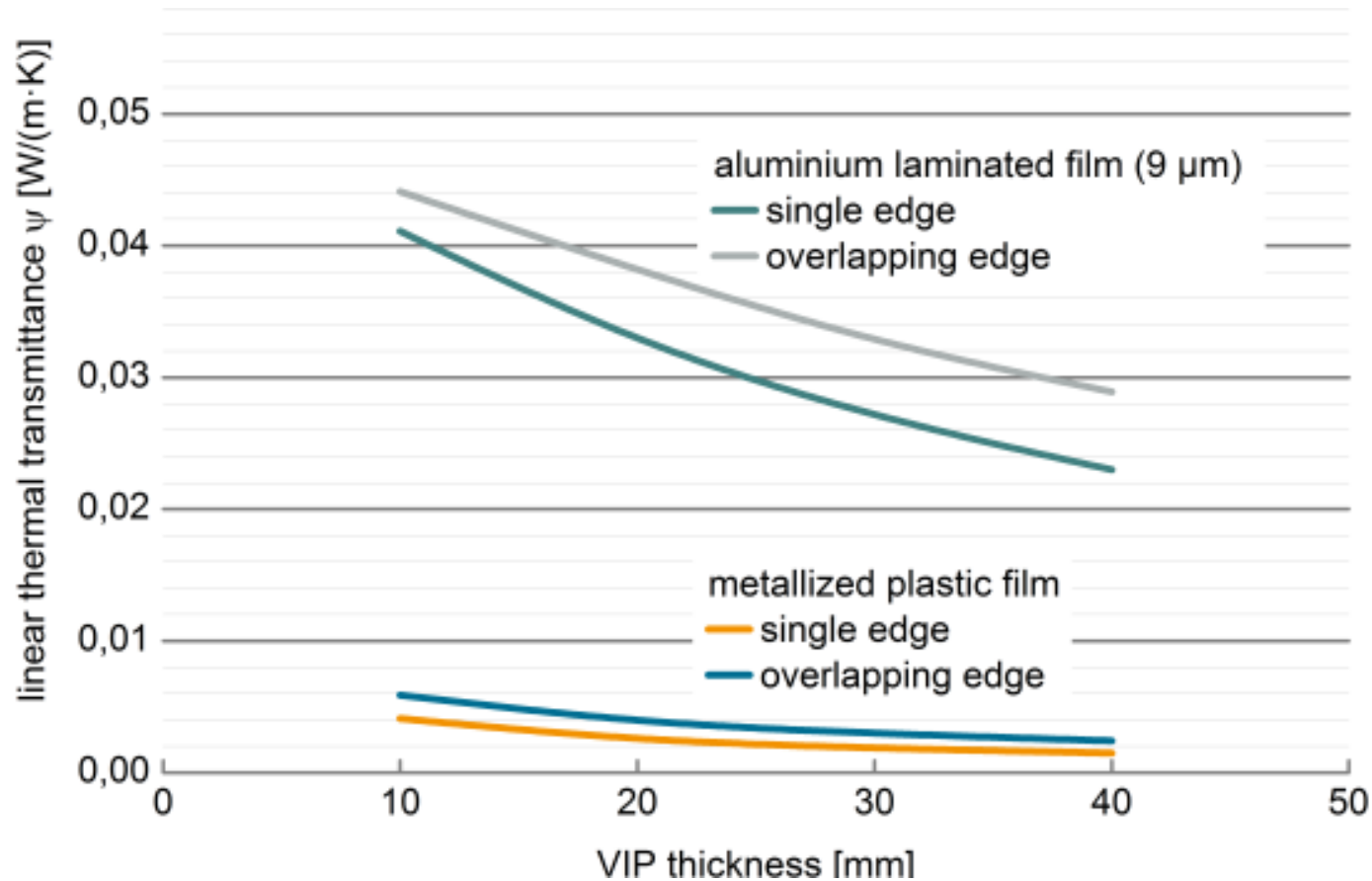


Photo: FIW

- Edge design
- Material and thickness of barrier layer
- Gap width between two panels and gap filler material
- Cover layer material
- Mounting and fixing
- 2-layered constructions

# Edge design and thickness of barrier layer





# Edge Design



Single layer edge design



overlapping edge design

*Graphics: FIW München*

# How „severe“ is $\psi$ ?

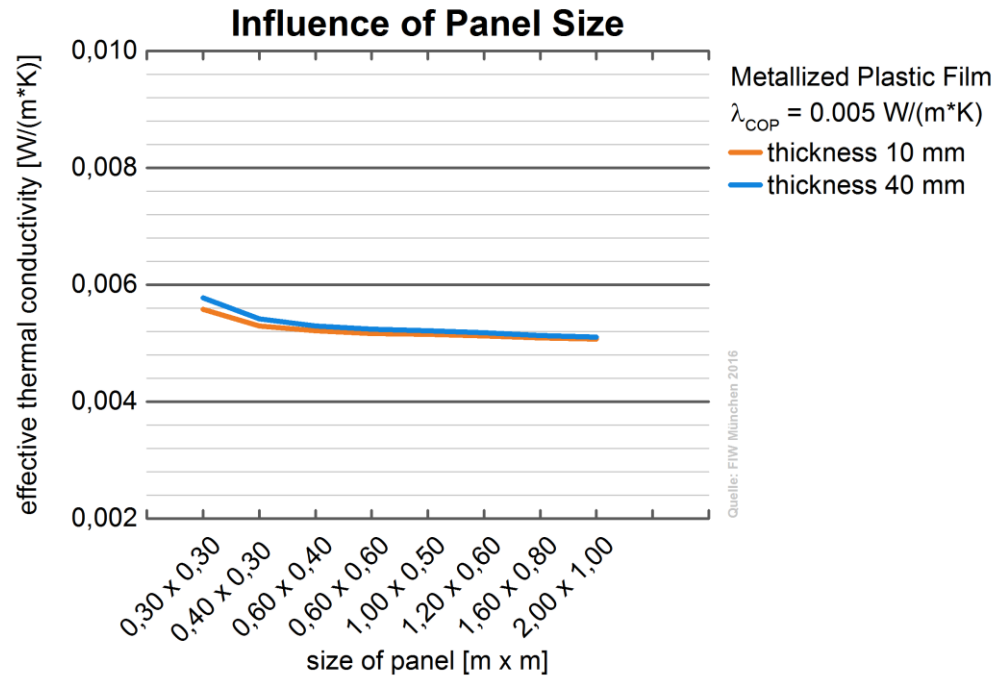
- Depending on the  $\psi$ -value
    - Aluminium laminated foils
  - vs.
  - Metallized plastic films
- Fluctuations in temperature difference
  - Increased local heat-flux
  - Lower R-value = higher efficient thermal conductivity



*Photo: variotec*

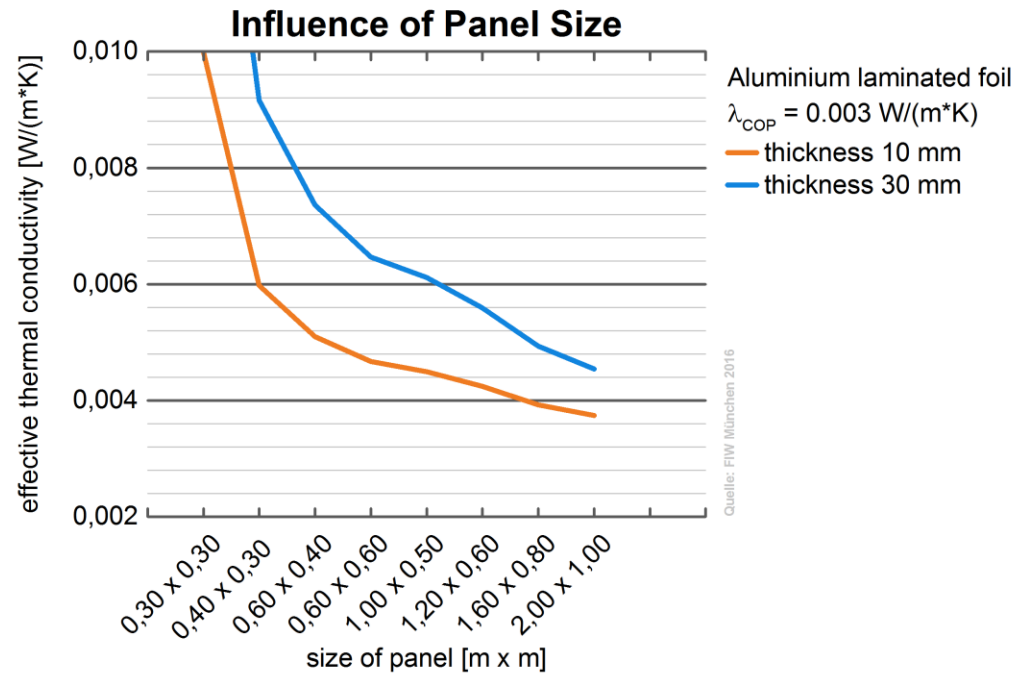
# How „severe“ is $\psi$ ? Effect on Lambda...

## Metallized plastic film



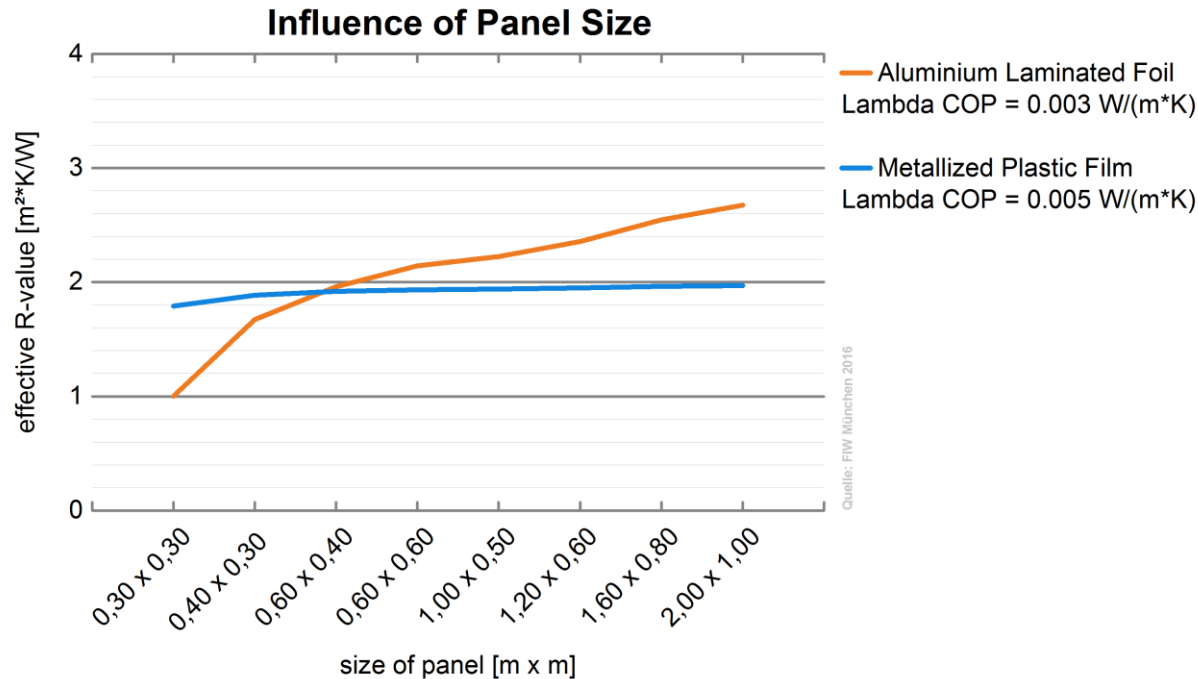
# How „severe“ is $\psi$ ? Effect on Lambda...

## Aluminium laminated foil



# How „severe“ is $\psi$ ? Effect on R-value...

## Comparison of 10 mm panels with silica and fiber glass cores



# Hybrid Envelope



Aluminum laminated film

Metallized plastic film

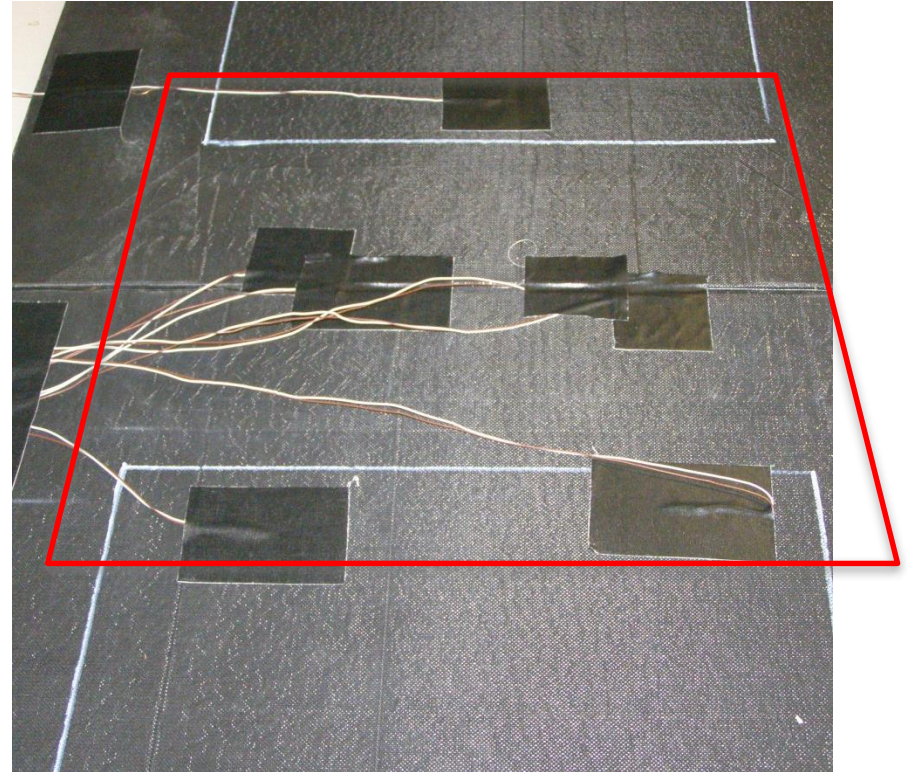
*Graphics: FIW München*



# Measurement of $\psi$ in GHP/HFM

# Determination of $\psi$ by measurement

- Significant joint length within the metering area
- Consideration of non-uniform temperature distribution
  - Sensors directly on the joint
  - Sensors in the slightly effected area
  - Sensors in the COP
- Temperature difference to be area weighted and averaged



*Thermal bridge measurement: single joint*

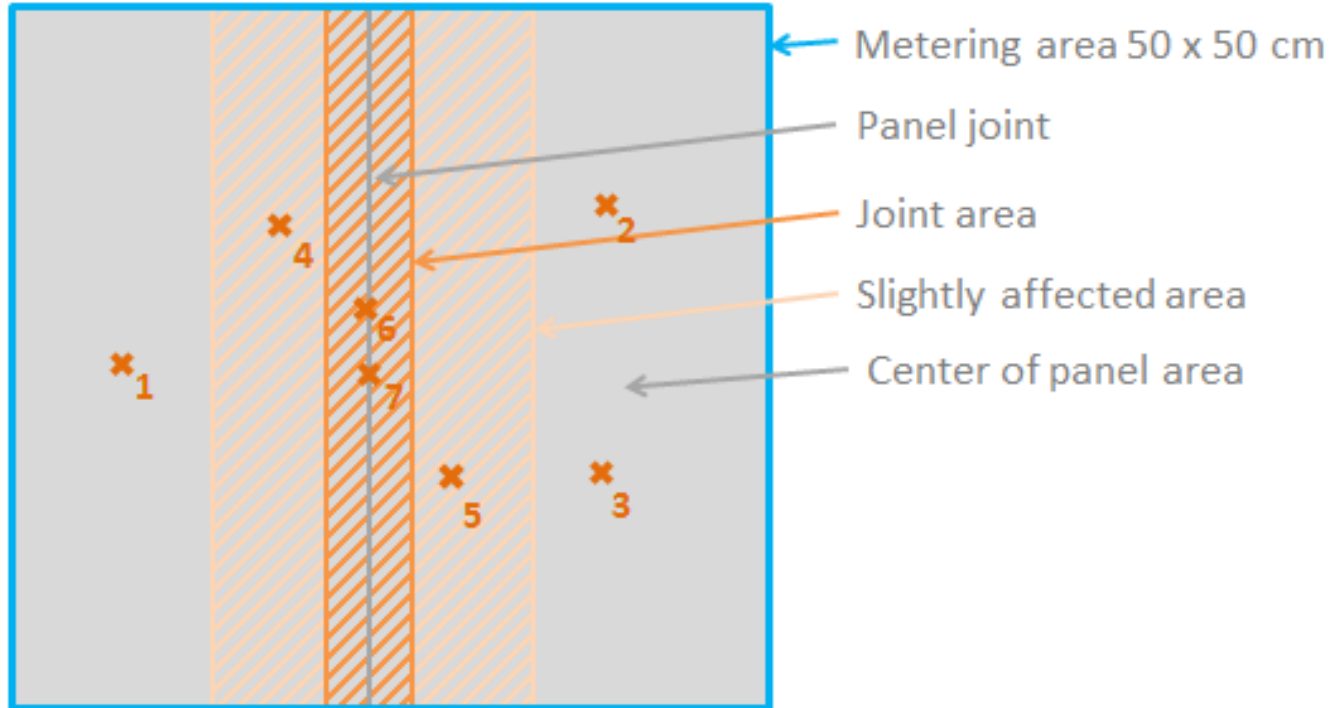


# Area weighted temperature difference

$$\Delta\theta_m = \frac{A_{COP} \cdot \Delta\theta_{COP} + A_{SA} \cdot \Delta\theta_{SA} + A_{joint} \cdot \Delta\theta_{joint}}{A_{COP} + A_{SA} + A_{joint}}$$

$\Delta\theta_m$	Area weighted temperature difference for joint assembly in K
$A_{COP}$	Center of panel area in m <sup>2</sup>
$A_{SA}$	Area slightly affected in m <sup>2</sup>
$A_{joint}$	Joint area (strongly affected) in m <sup>2</sup>
$\Delta\theta_{COP}$	Temperature difference for COP area in K
$\Delta\theta_{SA}$	Temperature difference for SA area in K
$\Delta\theta_{joint}$	Temperature difference for Joint area in K

# Positioning of Thermocouples



# GHP: Effect. Th. Cond. for joint assembly



$$\lambda_{eq\ ja} = \frac{\Phi \cdot d_{panel}}{A \cdot \Delta\vartheta_m}$$

$\lambda_{eq\ ja}$	equivalent thermal conductivity for joint assembly in W/(m·K)
$\Phi$	electrical power input for hot-plate metering area in W
$d_{panel}$	thickness of panel (equal thickness of joint assembly and COP specimens required for this method) in m
$A$	metering area in m <sup>2</sup>
$\Delta\theta_m$	area weighted temperature difference for Joint assembly in K

# HFM: Effect. Th. Cond. for joint assembly



$$\lambda_{eq\ ja} = \frac{d_{panel}}{R_{eq\ ja}}$$

$$R_{eq\ ja} = \frac{\Delta\theta_m}{q}$$

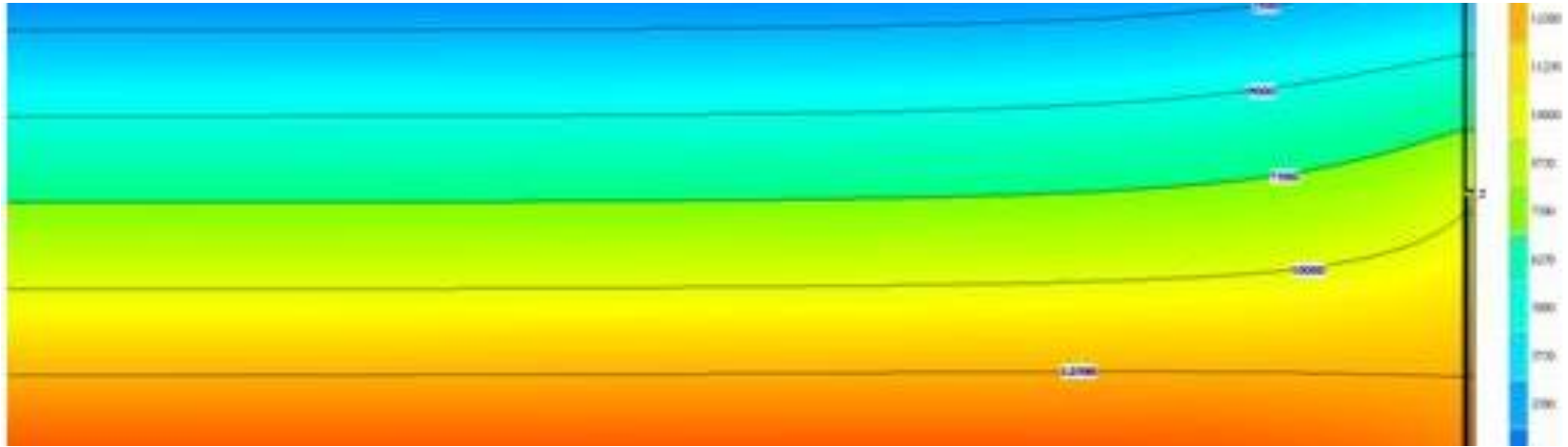
$\lambda_{eq\ ja}$	equivalent thermal conductivity for joint assembly in W/(m·K)
$q$	heat-flux density from HFM in W/m <sup>2</sup>
$d_{panel}$	thickness of panel (equal thickness of joint assembly and COP specimens required for this method) in m
$R_{eq\ ja}$	Equivalent thermal resistance for joint assembly in m <sup>2</sup> ·K/W
$\Delta\theta_m$	area weighted temperature difference for Joint assembly in K

# Determination of linear thermal transmittance



$$\psi = \frac{A}{d \cdot l_{\psi}} \cdot (\lambda_{eq\ ja} - \lambda_{COP})$$

$\psi$	linear thermal transmittance for the joints in the metering area in $W/(m \cdot K)$
$l_{\psi}$	length of the joints within the metering area in m
$\lambda_{eq\ ja}$	equivalent thermal conductivity including edge effects for the specific joint assembly in $W/(m \cdot K)$
$\lambda_{COP}$	thermal conductivity for center of panel in $W/(m \cdot K)$

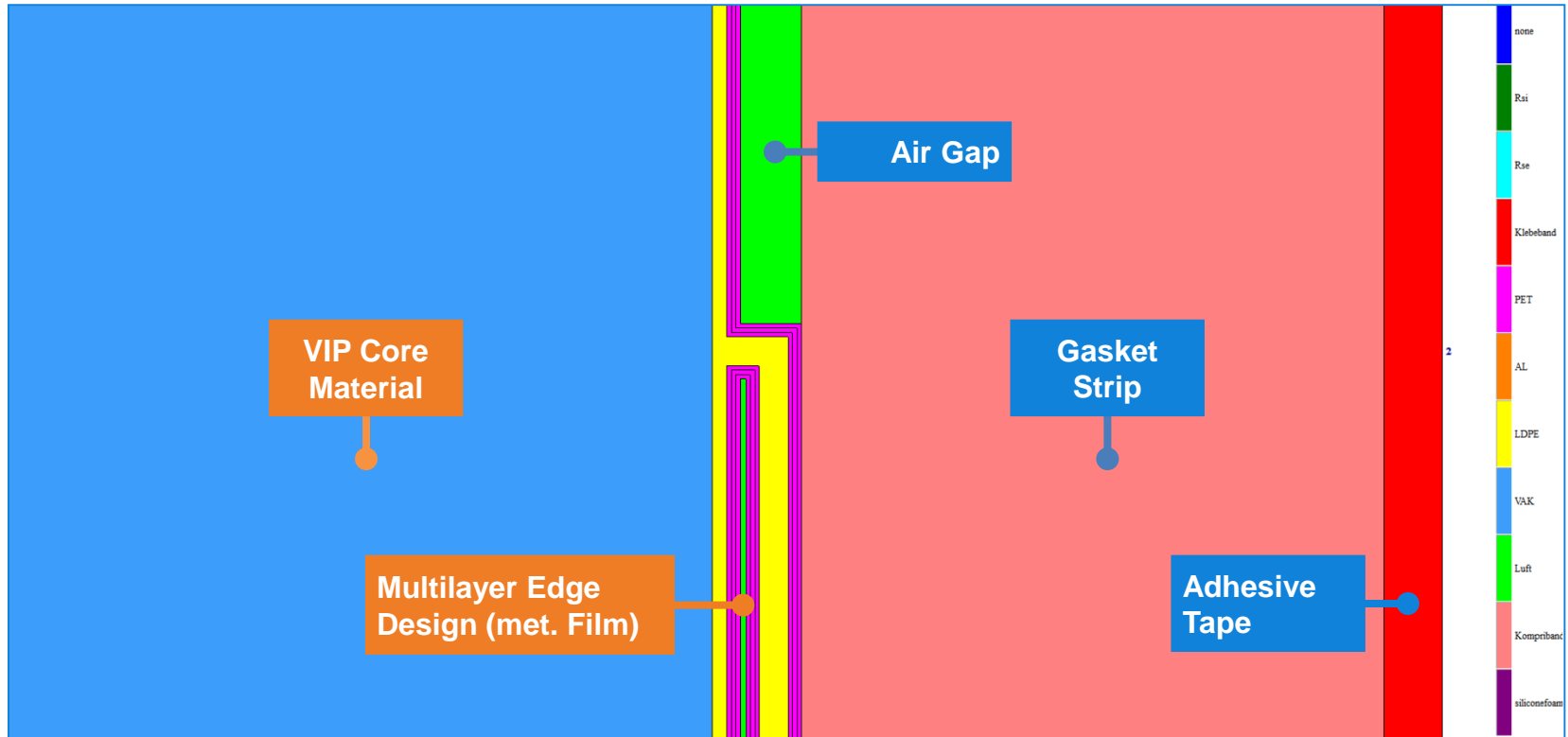


# Supporting FD-simulations

# Supporting FD-Simulations

- Width of influenced areas depend on
  - Cross conduction in
    - Barrier layers of foil
    - Cover layers of panel
    - Adjacent heating and cooling plates of apparatus
    - Joint filler material
    - Edge design etc.
  
- Dimensions of areas can be obtained by FD-Simulations

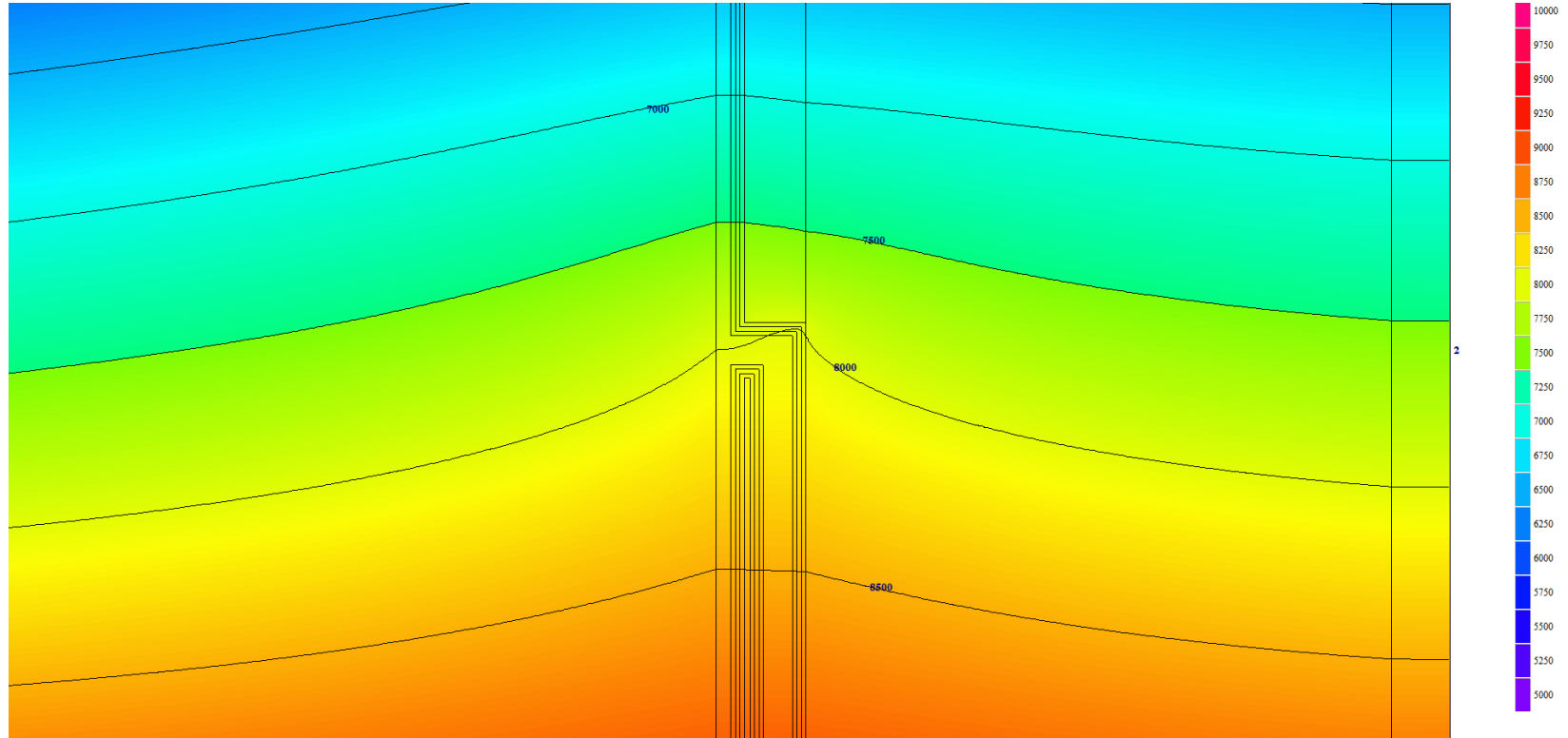
# Supporting FD-Simulations



*2D-Model for FD-Sim. of overlapping edge design with precompressed gasket strip and adhesive tape*

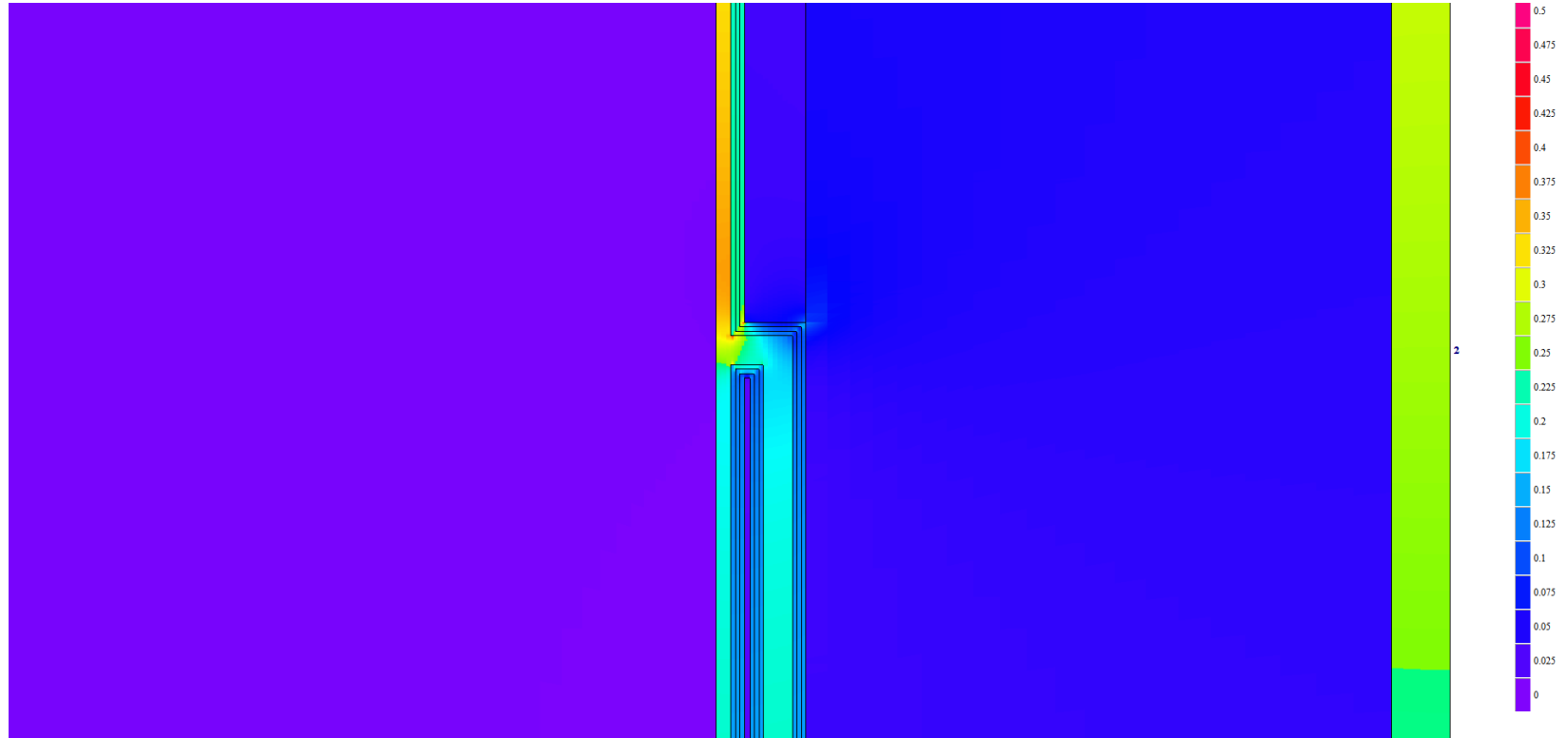


# Supporting FD-Simulations



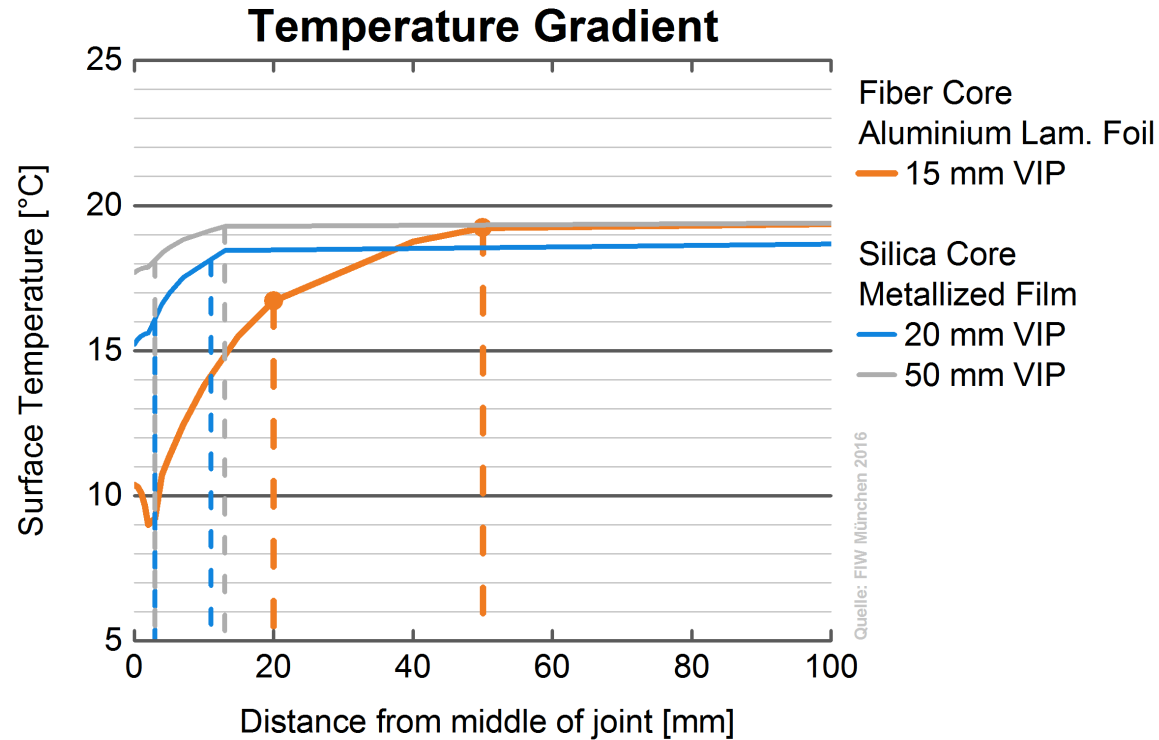
*Temperature distribution (5 to 10°C) of overlapping edge design with precompressed gasket strip and adhesive tape*

# Supporting FD-Simulations



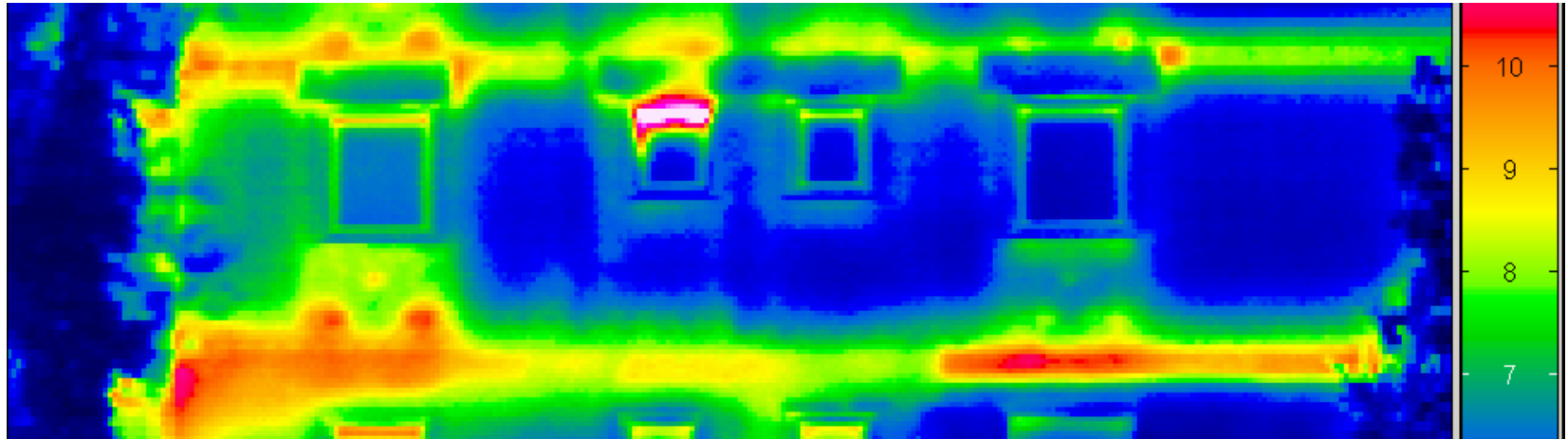
*Total heat-flux density of overlapping edge design with precompressed gasket strip and adhesive tape*

# Temperature Gradient at Panel Edge



# Area determination for 500 x 500 mm HFM

Panel	Joint Area	Slightly Affected Area	Center of Panel
FIB-AL.15	20 mm	30 mm	200 mm
FIB-MET.15	2 mm	15 mm	233 mm
FIB-HY.15	20 mm AL 2 mm MET	30 mm AL 15 mm MET	200 mm AL 233 mm MET
SIL-MET.20	2 mm	11 mm	237 mm
SIL-MET.50	2 mm	13 mm	235 mm



# Results and discussion

# Results

	FIB-AL	FIB-HY	FIB-MET	SIL-MET.20	SIL-MET.50
$\lambda_{COP}$ [W/(m·K)]	0.0020 $\pm 0.00008$	0.0021 $\pm 0.000084$	0.0022 $\pm 0.000088$	0.00435 $\pm 0.00014$	0.0050 $\pm 0.00015$
$\lambda_{eq\ ja}$ [W/(m·K)]	0.0056 $\pm 0.000448$	0.0032 $\pm 0.000256$	0.0027 $\pm 0.000216$	0.00498 $\pm 0.00035$	0.0062 $\pm 0.00043$
$\Psi_{measured}$ [W/(m·K)]	0.117 $\pm 0.00937$	0.037 $\pm 0.00296$	0.016 $\pm 0.00128$	0.015 $\pm 0.0012$	0.013 $\pm 0.00104$
$\Psi_{calculated}$ [W/(m·K)]	0.089	-	-	0.0144	0.0073
$\lambda_{eff}$ panel size 1.0m x 0.5 m [W/(m·K)]	0.0079	0.0039	0.0030	0.0054	0.0070
$\lambda$ increase from $\lambda_{COP}$ to $\lambda_{eff}$ [%]	293	85	35	23	39

# Results and Discussion

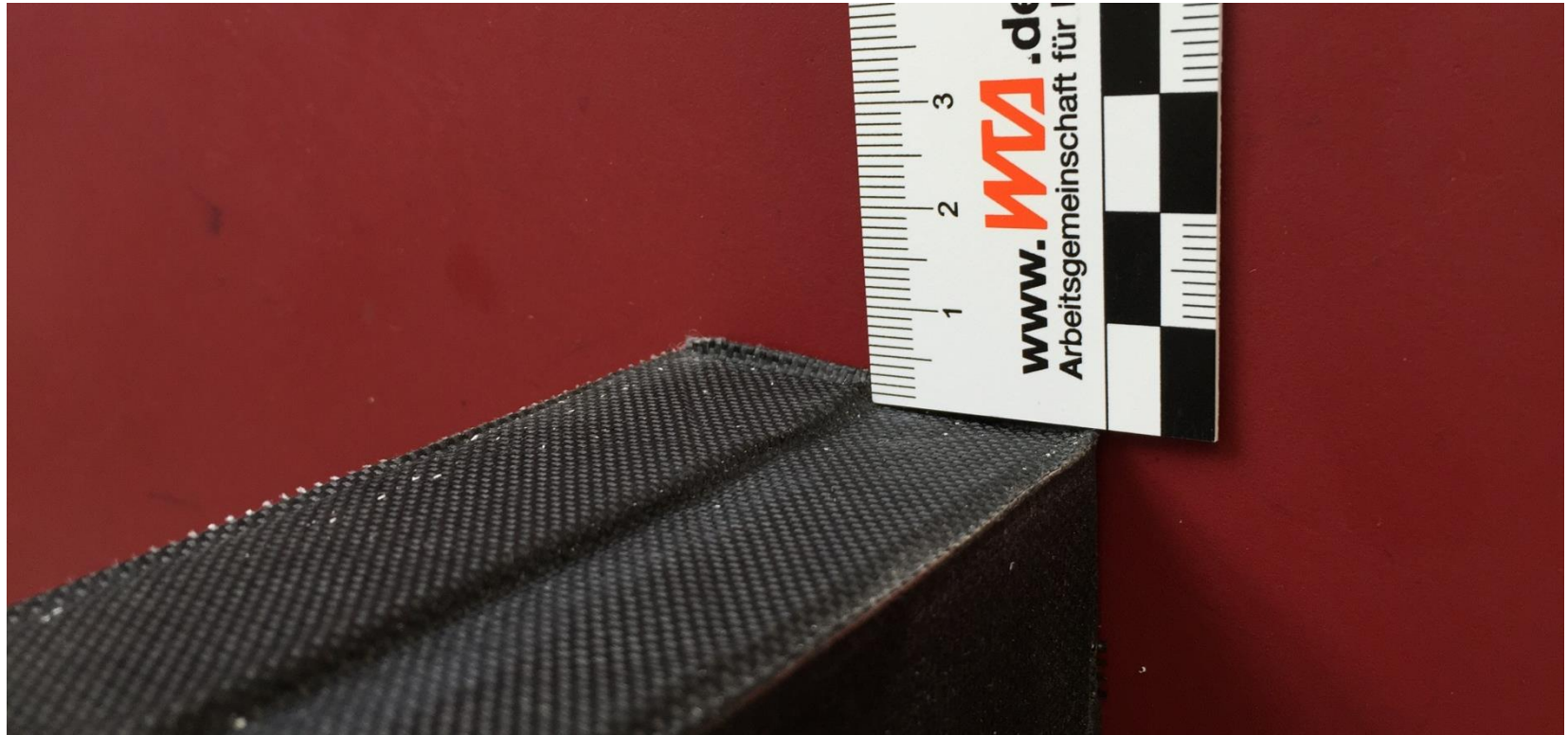
- Huge influence from thermal bridges on  $\lambda_{\text{eff}}$ !
  - Silica core with metallized film: 10 to 40 %
  - Fiber core
    - Aluminium foil: 20 to 300 %
    - Hybrid solution: 15 to 100 %
    - Metallized Film: 10 to 40 %
  
- Aluminium Foil recommend for „bigger“ panels
  
- Hybrid solutions with getters and/or desiccants

# Results and Discussion

- Good agreement for 20 mm VIP
  - 5% - 8% uncertainty for HFM
  
- Measurement for 50 mm VIP is significantly higher than  $\psi$  from numerical simulation
  - Even when considering extended uncertainty of 8% - 10%
  - Increased  $\lambda_{\text{COP}}$  for one VIP
  - Width of the joint significantly bigger
  - Edges of the panels uneven
  - Made from stacked core material slabs

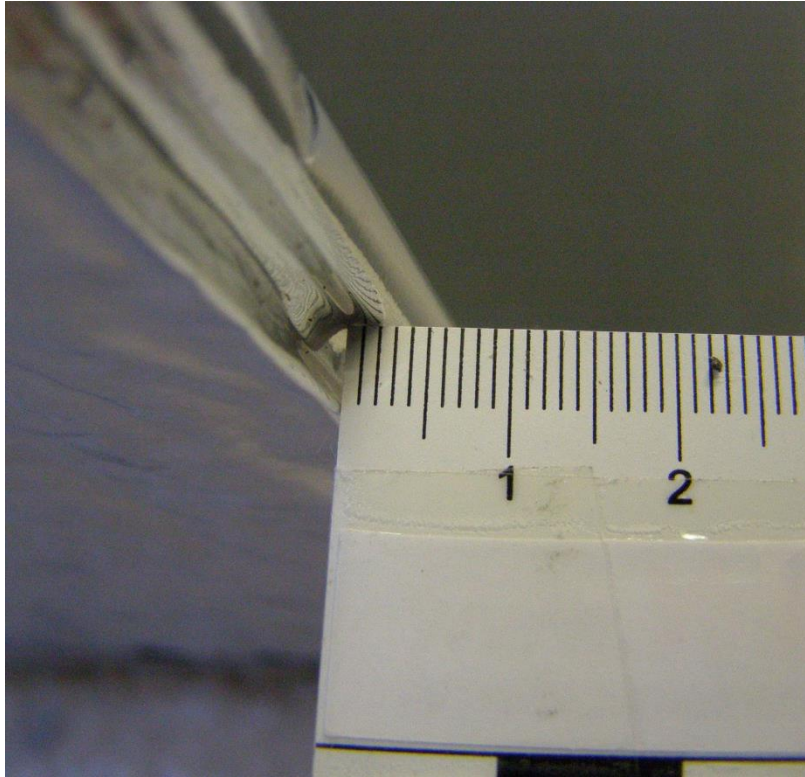


# Results and Discussion

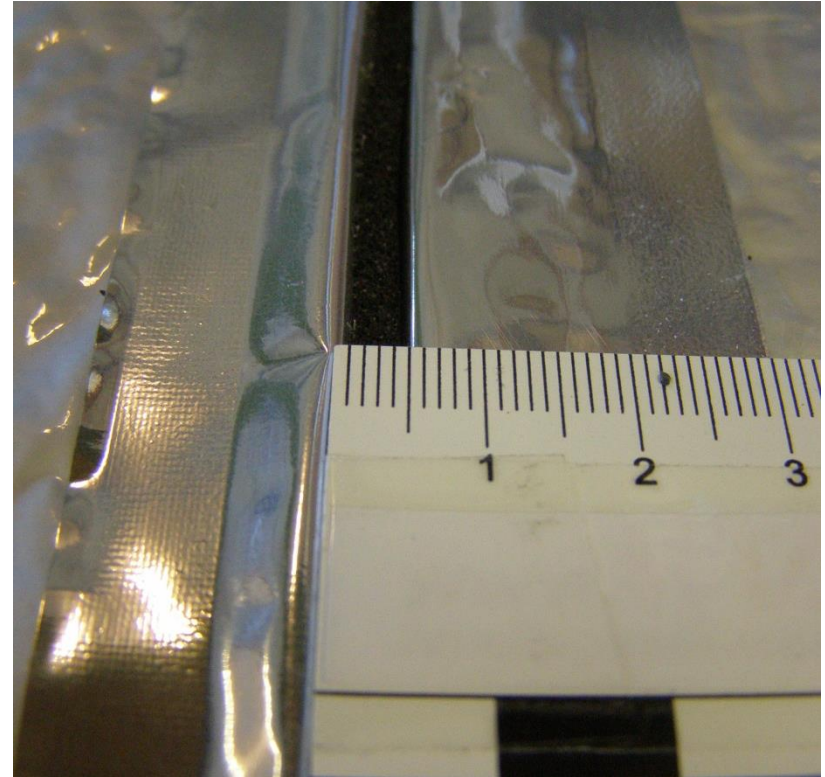


*Uneven edge with additional 2 mm offset*

# Results and Discussion



*Photo: FIW München*



*Photo: FIW München*

# Results and Discussion



- Recommendations for assembly and panel size
  - Big enough to ensure significant COP area within metering area
  - Reasonable joint length
  - Ensure tight mounting
  - Ensure air-tightness
  - Ensure good contact of heating and cooling plates
  - Measure COP of both adjacent panels
  - Use contact layers in GHP or HFM measurement
  - When using GHP: make sure your regulation of the ring is o.k.

# Results and Discussion

- Accurate measurement of  $\psi$  is not easy
- Depending on a large variety of influencing factors
- Even small deviations in the measurement setup can lead to large uncertainties in the result (differential measurement)
- Exact definition of boundary conditions needed, if measurement of  $\psi$  shall be used for the determination of thermal performance of VIP
  - Distance between panels
  - Joint filler material
  - Influence areas for temperature difference etc.
- Recommend to use numerical simulations with FD-Method instead of measurements



# Thank You!



**Christoph Sprengard**  
FIW München  
Lochamer Schlag 4  
82166 Gräfelfing  
Phone +49 89 85800-58, Fax -40  
[sprengard@fiw-muenchen.de](mailto:sprengard@fiw-muenchen.de)