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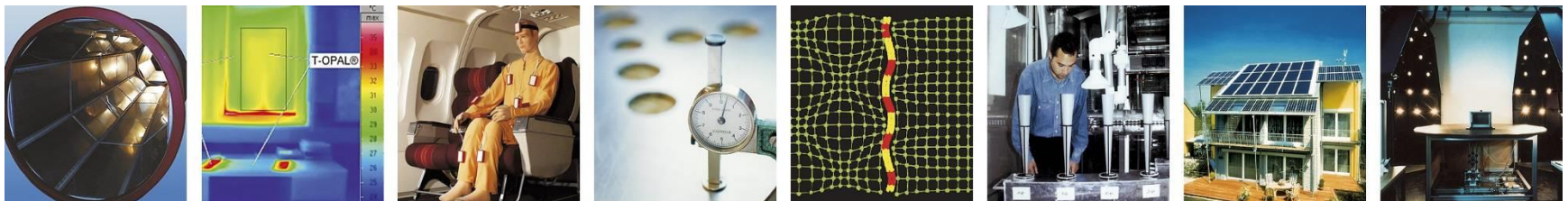
# Implementation and validation of a long-wave heat exchange model

Matthias Winkler, Florian Antretter and Jan Radon

Buildings XIII – Clearwater Beach, FL – 2016-12-08

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Building on knowledge



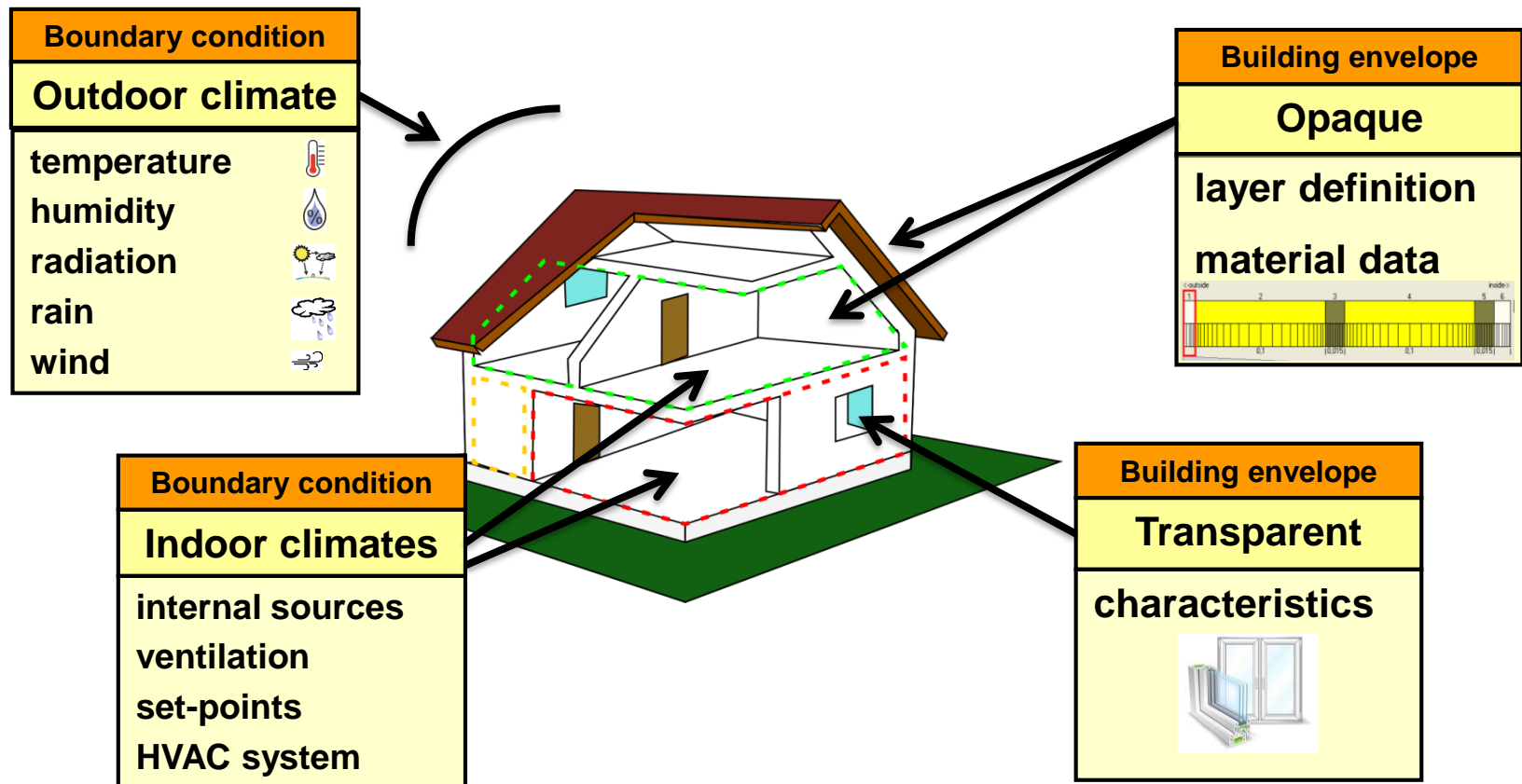
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# Agenda

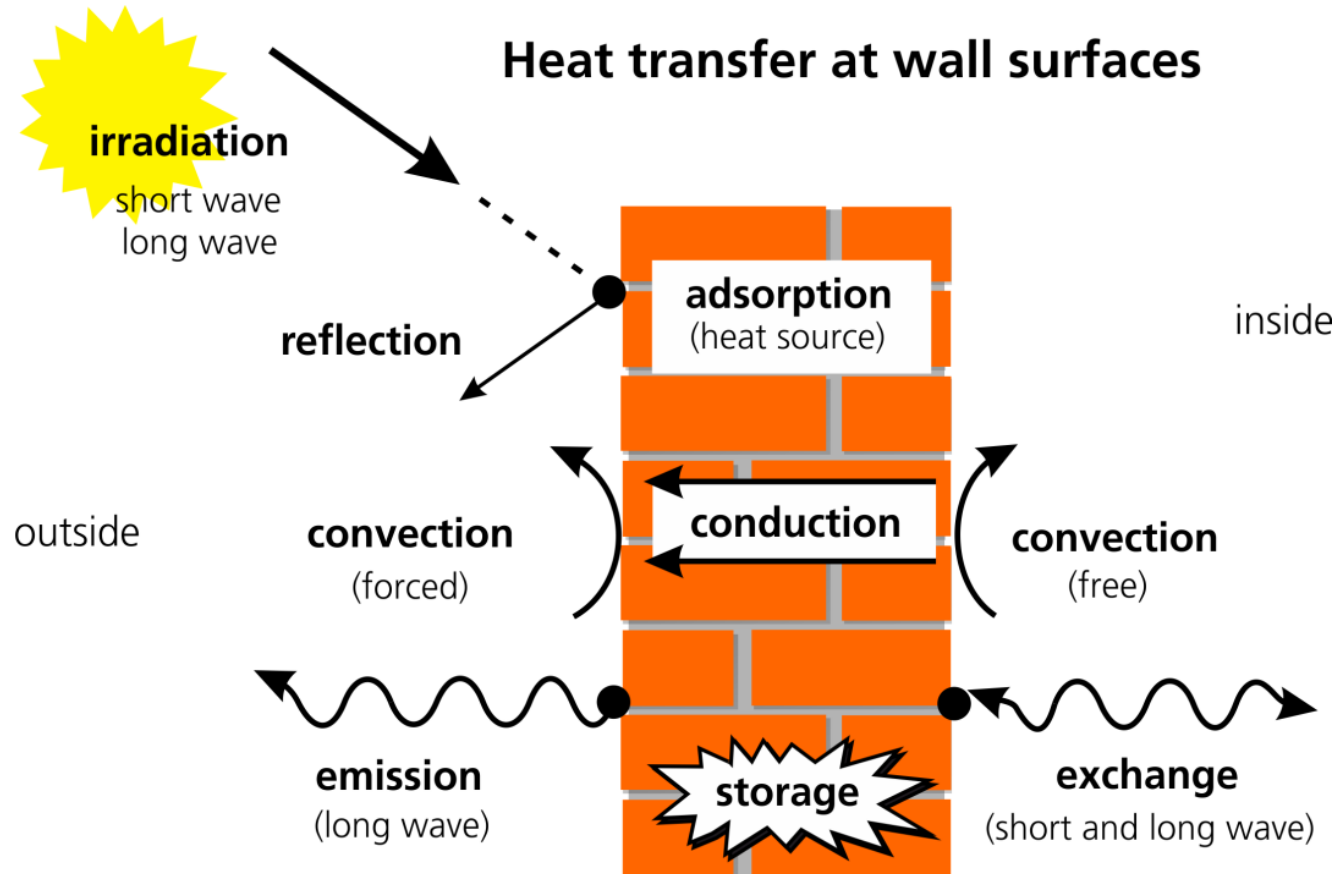
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- Current Approach in WUFI® Plus
- Inner Longwave Heat Exchange Model
- Validation
- Application
- Outlook
- Conclusion

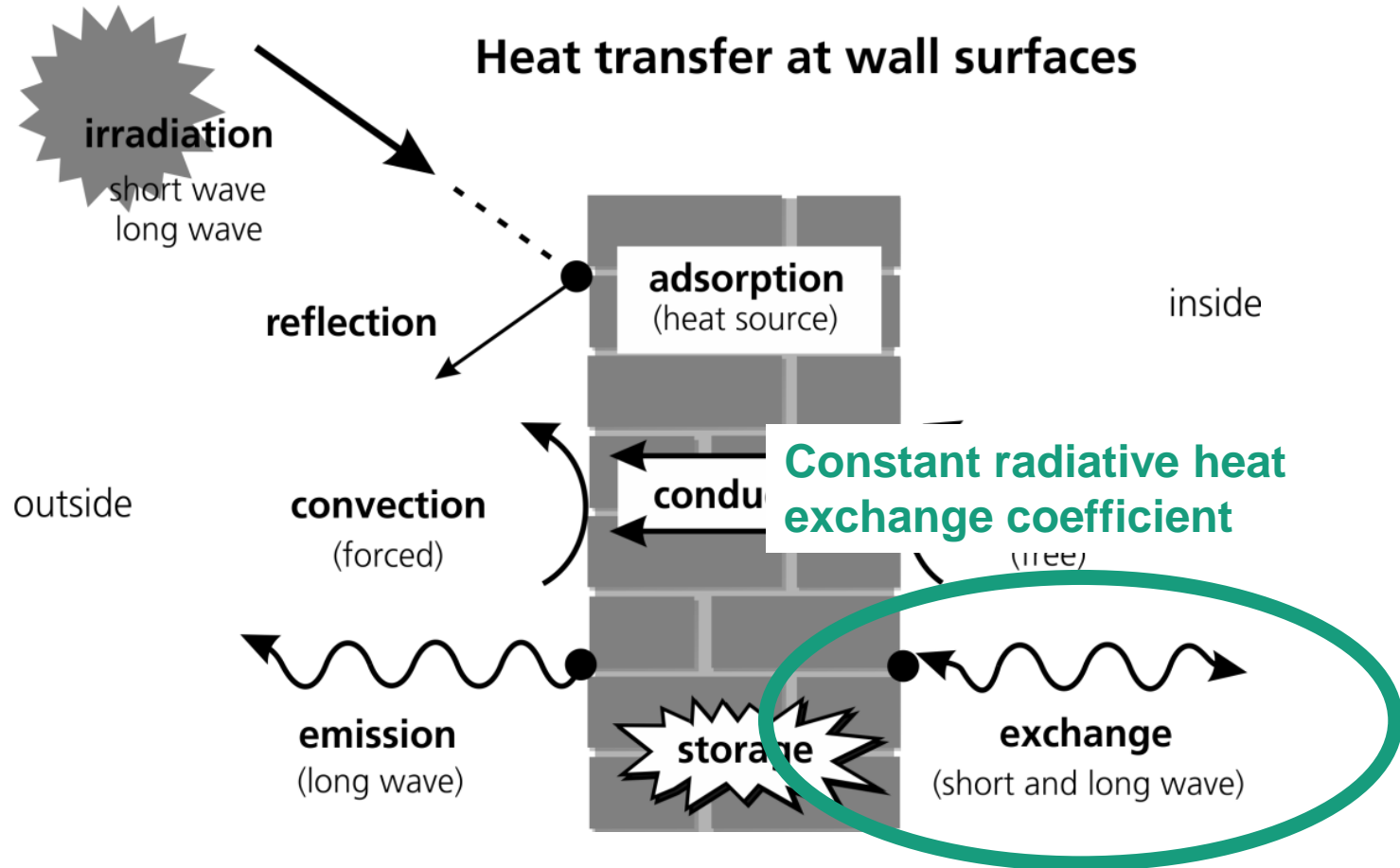
# Hygrothermal Building Simulation with WUFI® Plus



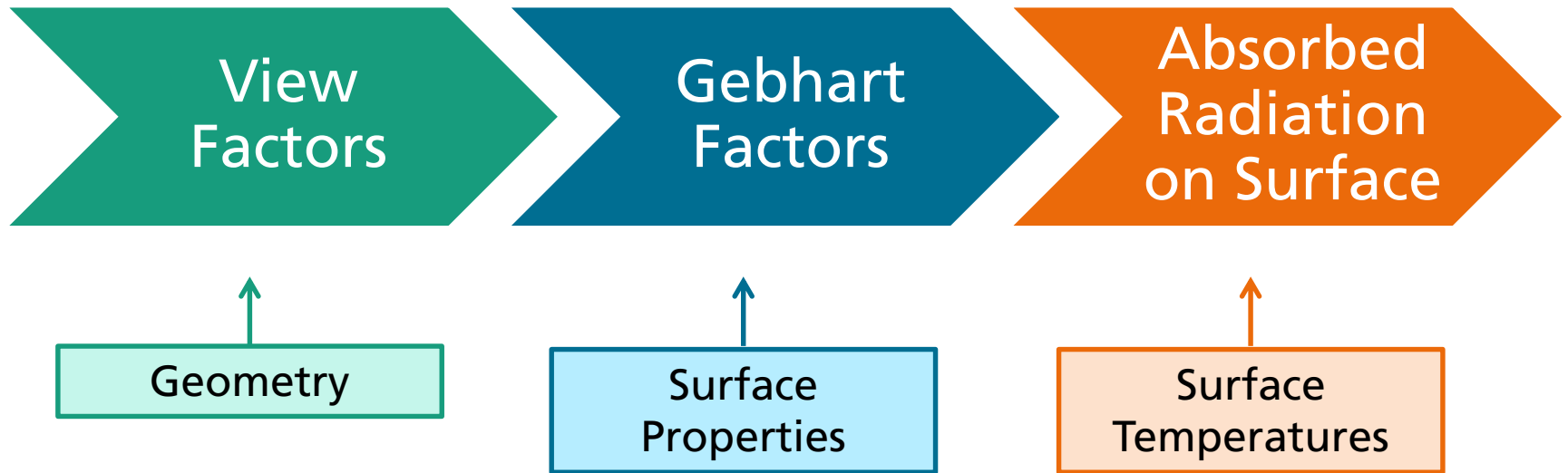
# Hygrothermal Building Simulation with WUFI® Plus



# Hygrothermal Building Simulation with WUFI® Plus



# Calculation Method



# Calculation Method: View Factor

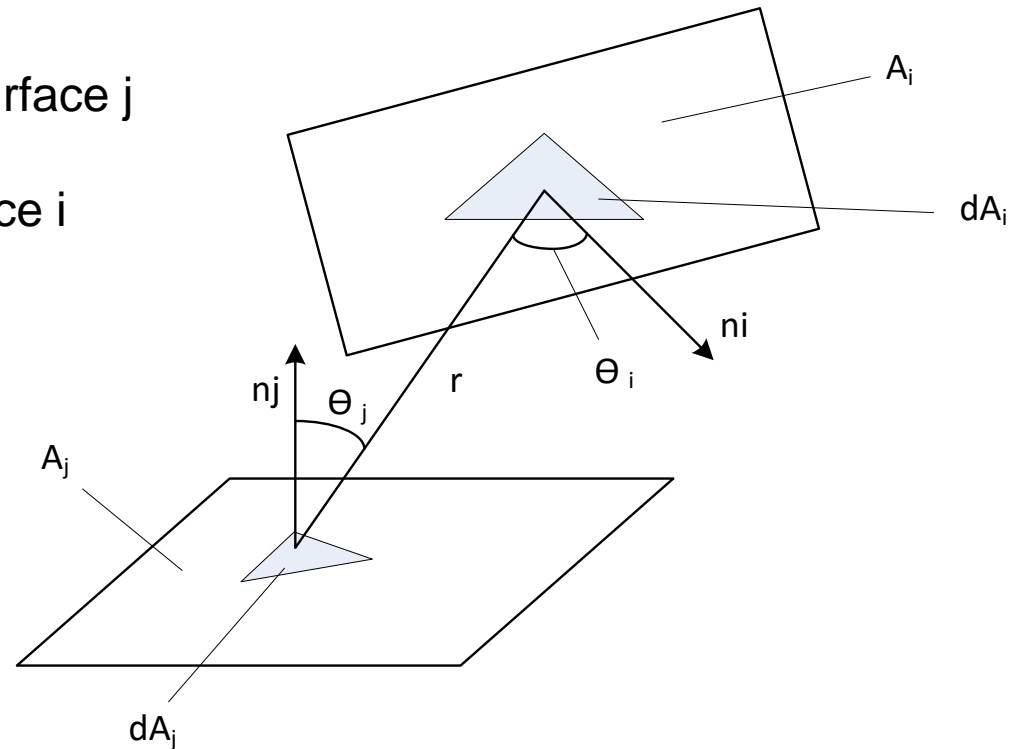
## ■ View Factor Definition:

- Fraction of rays emitted from surface  $i$  with direct incidence on surface  $j$  to all rays emitted from surface  $i$

## ■ Geometric relationship

- Solved before energetic calculations

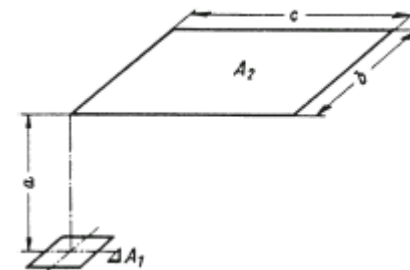
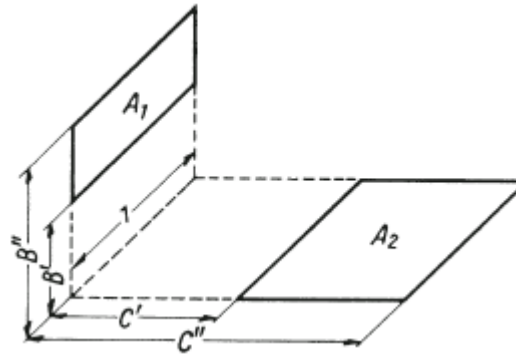
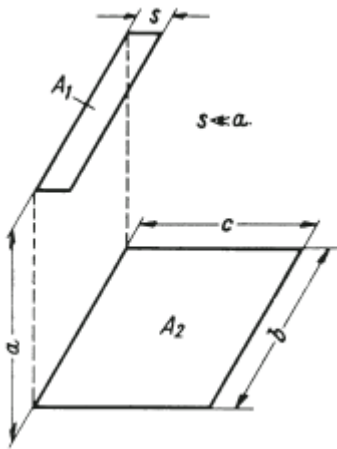
$$F_{ij} = \frac{1}{A_i} \iint_{A_i A_j} \frac{\cos \theta_i \cdot \cos \theta_j}{\pi \cdot r^2} \cdot dA_i \cdot dA_j$$



# Calculation Method: View Factor

Analytical solutions available, **but**

- limited number of shapes & positions
- Complicated / impossible analytical solutions for
  - more complex geometries
  - overshadowing



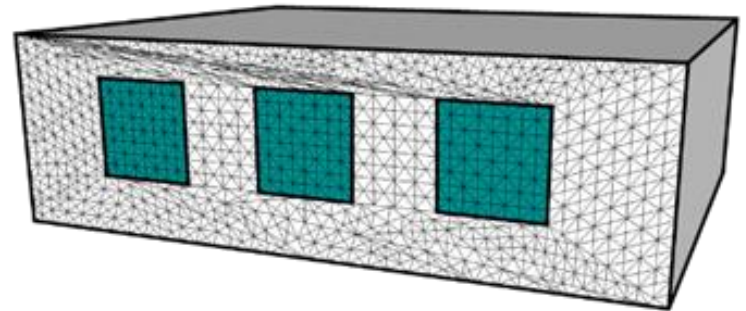
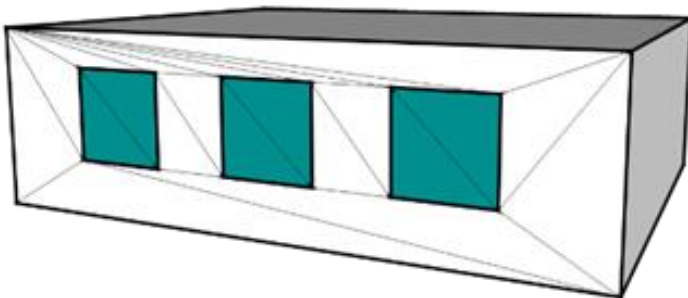
Images: VDI Wärmeatlas



# Calculation Method: View Factor

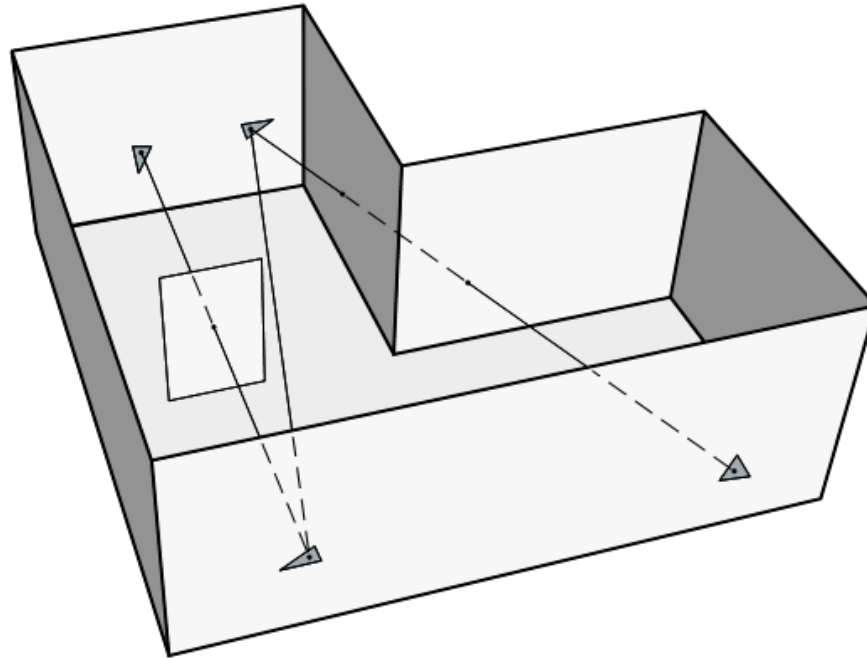
## → Solution: Triangulation

- Each surface divided into small triangles
- Calculation accuracy depends on triangle size
- All visualized components included



# Calculation Method: View Factor

- Connecting lines for every pair of triangles
- Vector analysis to check possible crossings
- Overshadowing from every visualized geometry



# Calculation Method: View Factor

- View Factor as fraction of connected triangles

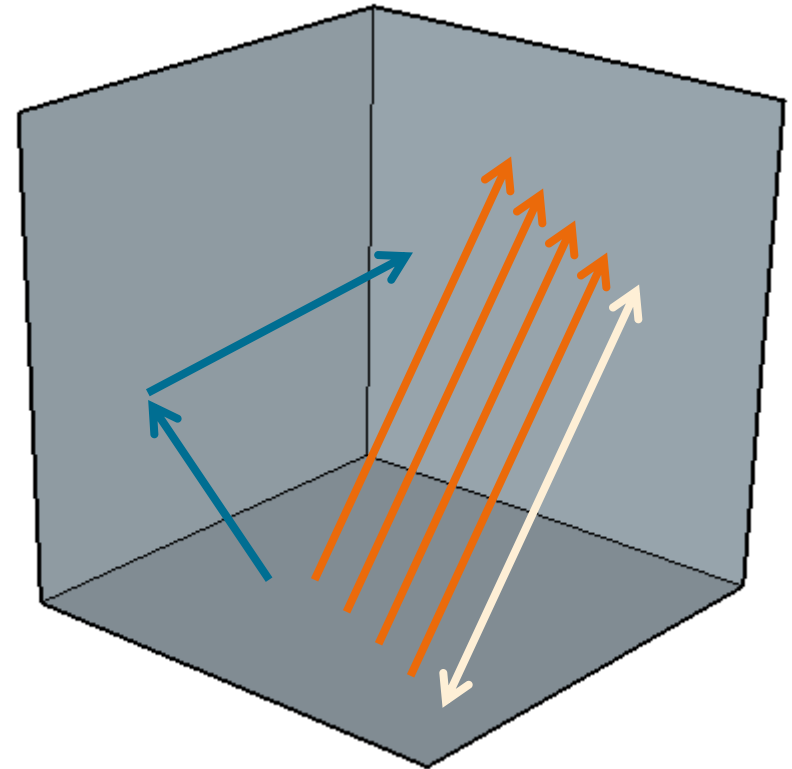
$$F_{ij} = \frac{1}{A_i} \cdot \sum_{i=1}^n \sum_{j=1}^m \frac{\cos \theta_i \cdot \cos \theta_j}{\pi \cdot r^2} \cdot dA_i \cdot dA_j$$

- Area weighting:
  - Sum of view factors for each surface must be 1.00
  - No exact numerical solution
  - Area-weighted distribution of remaining difference between numerical and exact solution

# Calculation Method: Gebhart Factor

- Gebhart Factor
  - Fraction of energy emitted from one surface that is absorbed at another surface
  - Considers all radiation paths, including multiple and own reflections
- Calculated from the View Factor
- Includes reflectivity and emissivity of all surfaces

$$G = (1 - F \cdot \rho)^{-1} \cdot F \cdot \varepsilon$$



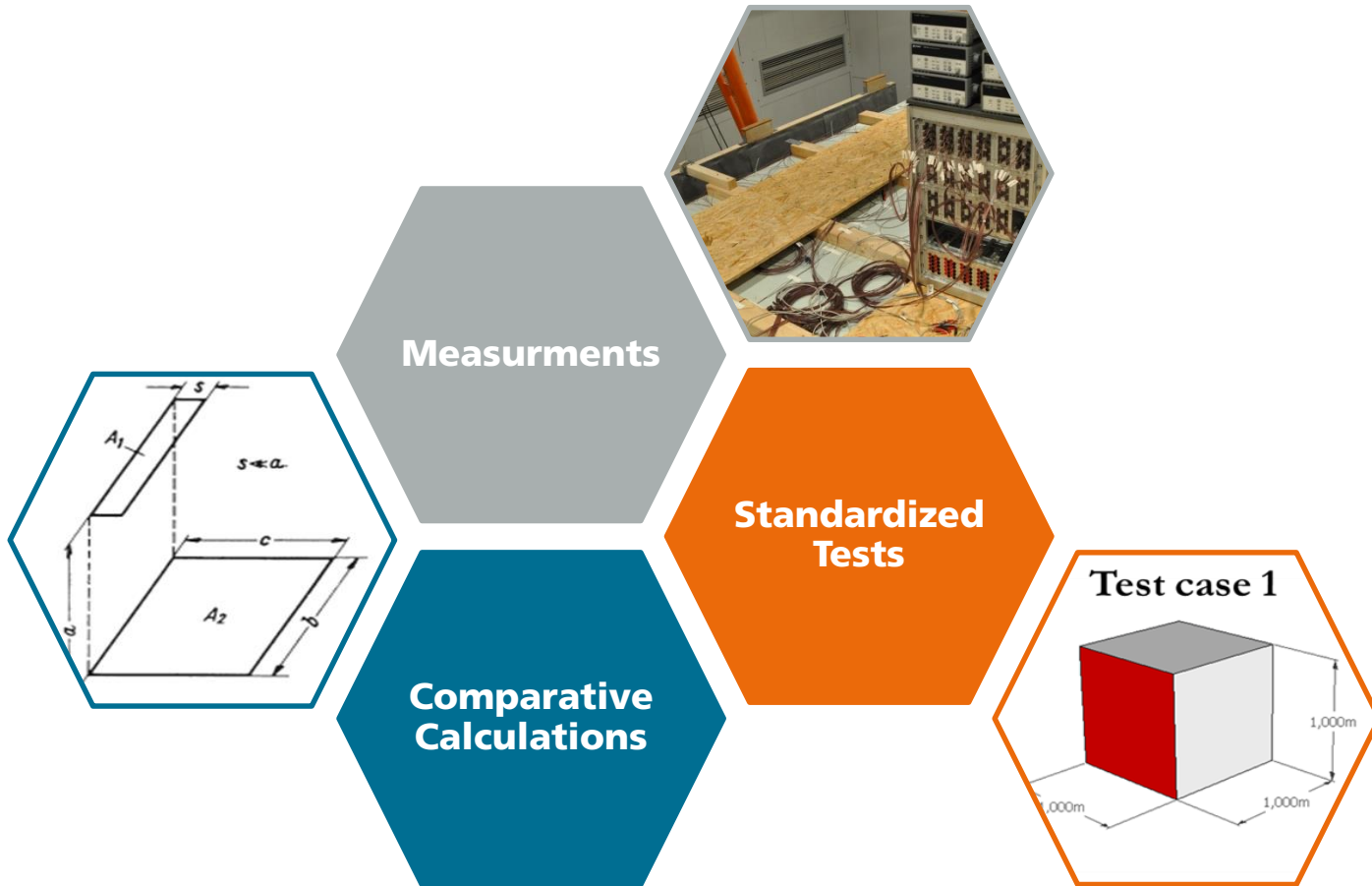
# Calculation Method: Longwave Radiation

- Longwave Radiation absorbed on an interior surface:

$$I_{l,i} = \varepsilon_i \cdot \sum_{j=1}^N G_{ij} \cdot \sigma \cdot (T_i^4 - T_j^4)$$

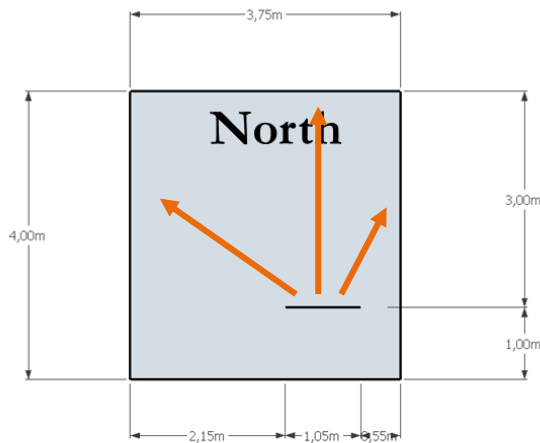
- Emissivity
- Gebhart-Factor
- Stefan-Boltzmann constant
- Temperatures of interior surfaces

# Validation

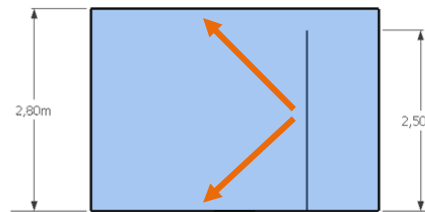


# Validation: View Factors (Exemplary Room)

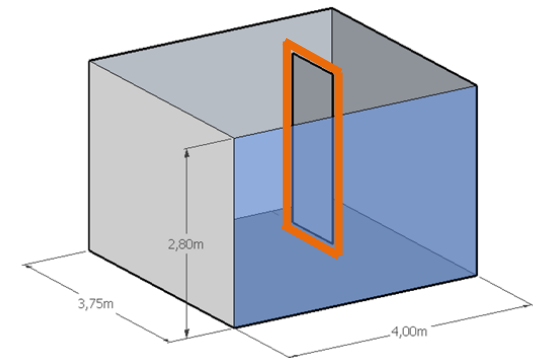
- Rectangular room with interior wall
  - Analytical solution (Howell, 2015)
  - WUFI® Plus calculation:
    - Accuracy levels = density of triangulation network



(a) Footprint

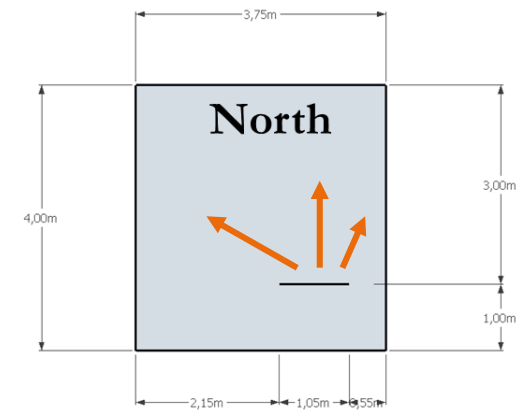


(b) Section



(c) Projection

# Validation: Exemplary Room



	Analytical	WUFI® Plus		
		Low accuracy	Medium accuracy	Very high accuracy
<b>F<sub>1,ceiling</sub></b>	0,2104	0,2172 (+3.23 %)	0,2127 (+1.09 %)	0,2105 (+0.05 %)
<b>F<sub>1,wall west</sub></b>	0,0787	0,0795 (+1.02 %)	0,0788 (+0.13 %)	0,0786 (-0.13 %)
<b>F<sub>1,floor</sub></b>	0,2538	0,2619 (+3.19 %)	0,2618 (+3.15 %)	0,2549 (+0.43 %)
<b>F<sub>1,wall north</sub></b>	0,2308	0,2324 (+0.69 %)	0,2312 (+0.17 %)	0,2308 (±0.00 %)
<b>F<sub>1,wall east</sub></b>	0,2262	0,2353 (+4.02 %)	0,2285 (+1.02 %)	0,2264 (+0.09 %)
<b>Sum</b>	<b>0,9999</b>	<b>1,0263</b> (+2.64 %)	<b>1,0129</b> (+1.30 %)	<b>1,0012</b> (+0.13 %)

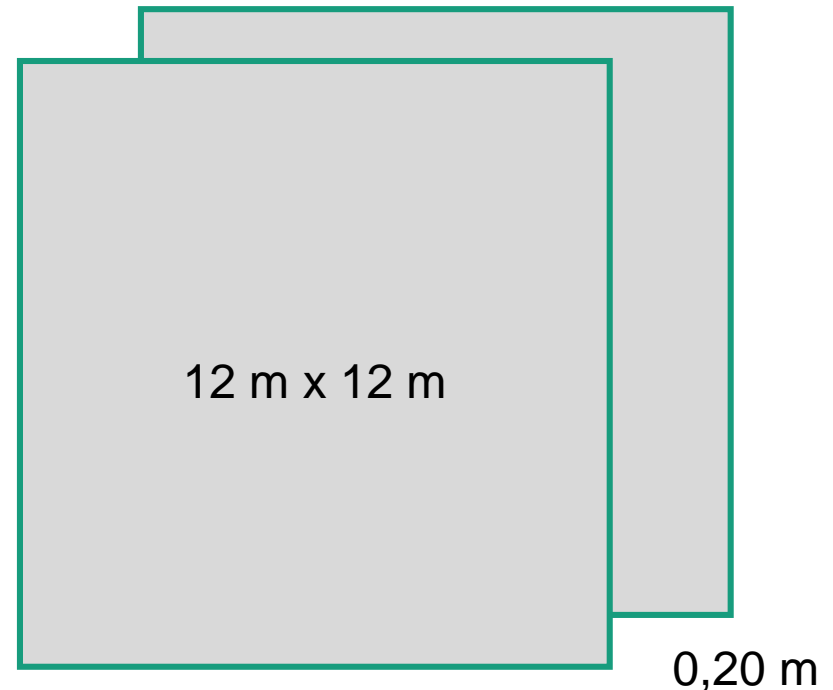


# Validation: Longwave radiation two parallel plates

- Analytic solution available
  - Includes emission, reflection & absorption
  - Neglects side-exchanges

$$q_{1,2} = \frac{1}{\frac{1}{\varepsilon_1} + \frac{1}{\varepsilon_2} - 1} \cdot \sigma \cdot (T_1^4 - T_2^4)$$

- Variations in boundary conditions:
  - Surface temperatures
  - Surface emissivity's



# Validation: Longwave radiation two parallel plates

		<b>Test 1</b>	<b>Test 2</b>	<b>Test 3</b>	<b>Test 4</b>
Surface Temperature 1	°C	50.00	50.00	30.00	30.00
Surface Temperature 2	°C	10.00	10.00	20.00	20.00
Emissivity surface 1	-	0.90	0.90	0.90	0.90
Emissivity surface 2	-	0.90	0.10	0.90	0.10
Longw. Heat Exchange: Analytical	W/m <sup>2</sup>	207.69	25.11	49.19	5.95
Longw. Heat Exchange: WUFI® Plus	W/m <sup>2</sup>	208.29	25.11	49.34	5.95
Deviation	%	<b>+ 0.29 %</b>	<b>+ 0.04 %</b>	<b>+ 0.30 %</b>	<b>+ 0.03 %</b>

# Validation: DIN EN ISO 13791

- DIN EN ISO 13791
  - Validation procedures for thermal building simulation software
  - Includes test cases for inner longwave heat exchange
  
- Test cases
  - 4 room-geometries with only opaque surfaces
  - Fixed boundary conditions
  - Validation criteria: Interior temperature

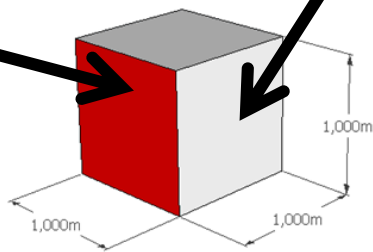
# Validation: DIN EN ISO 13791

**Exterior  
Wall**

**$T_e = 30\text{ °C}$**

**Absorption,  
Inner Side  
 $100\text{ W/m}^2$**

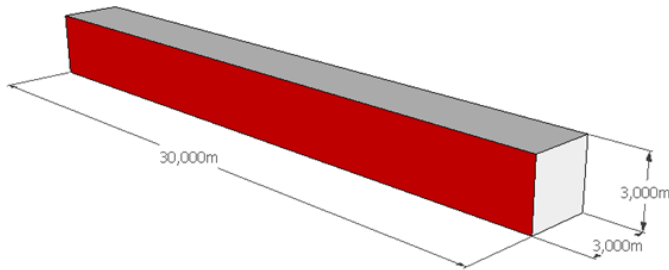
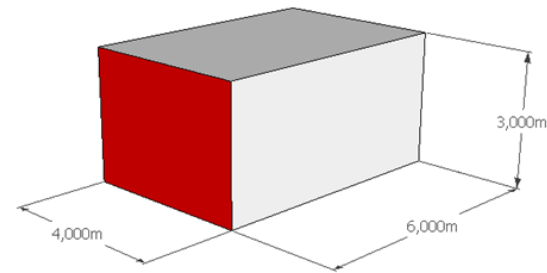
**Test case 1**



**Partition  
Walls**

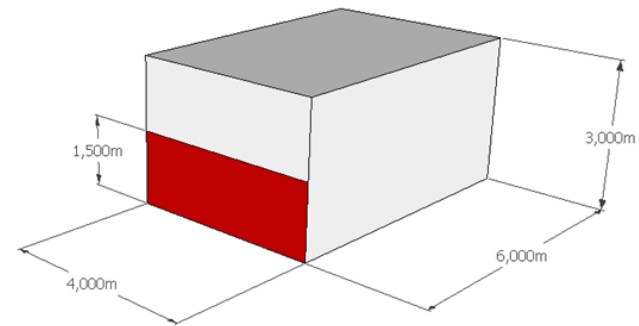
**$T_e = 20\text{ °C}$**

**Test case 2**



**Test case 3**

**Test case 4**



# Validation: DIN EN ISO 13791

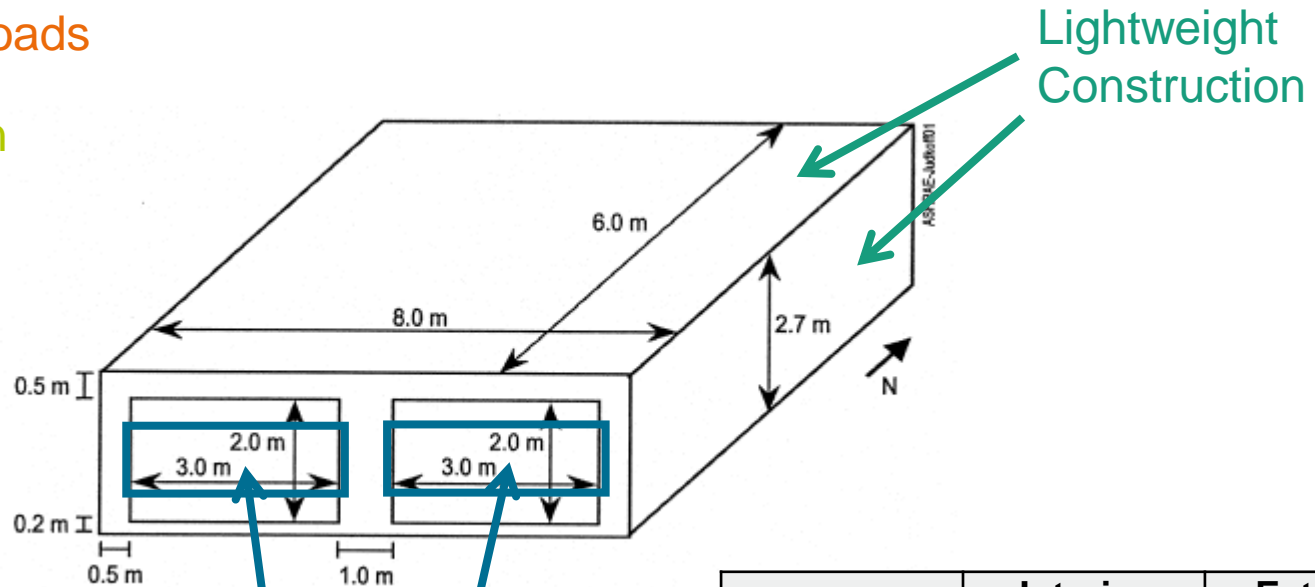
		Test 1	Test 2	Test 3	Test 4
<b>DIN EN 13791</b>	°C	34.4	30.4	38.5	25.5
<b>WUFI® Plus</b>	°C	34.3	30.1	38.3	25.0
<b>Deviation</b>	<b>K</b>	<b>0.1 ✓</b>	<b>0.3 ✓</b>	<b>0.2 ✓</b>	<b>0.5 ✓</b>

- Validation criteria:
  - Maximum allowed deviation: 0.5 K
  - Validation according to DIN EN 13791 successful

# Validation: ASHRAE 140 - Class I Test Procedures

No Internal Loads

No Ventilation

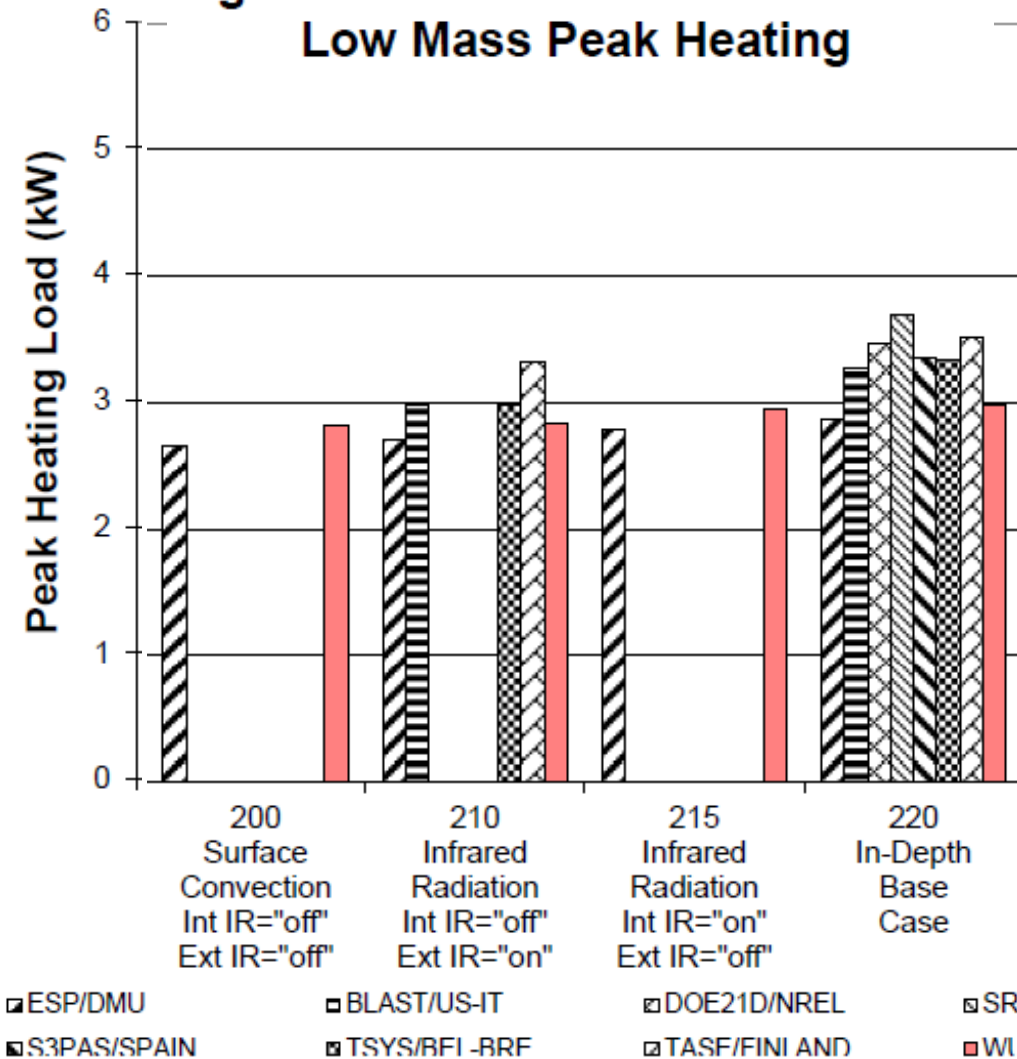


High-Conductance Wall

	Interior Longwave Emissivity	Exterior Longwave Emissivity
<b>Case 220</b>	0.9	0.9
<b>Case 215</b>	0.9	0.1
<b>Case 210</b>	0.1	0.9
<b>Case 200</b>	0.1	0.1

# Validation: ASHRAE 140 (Example Results)

Figure B8-35. BESTEST IN-DEPTH  
Low Mass Peak Heating



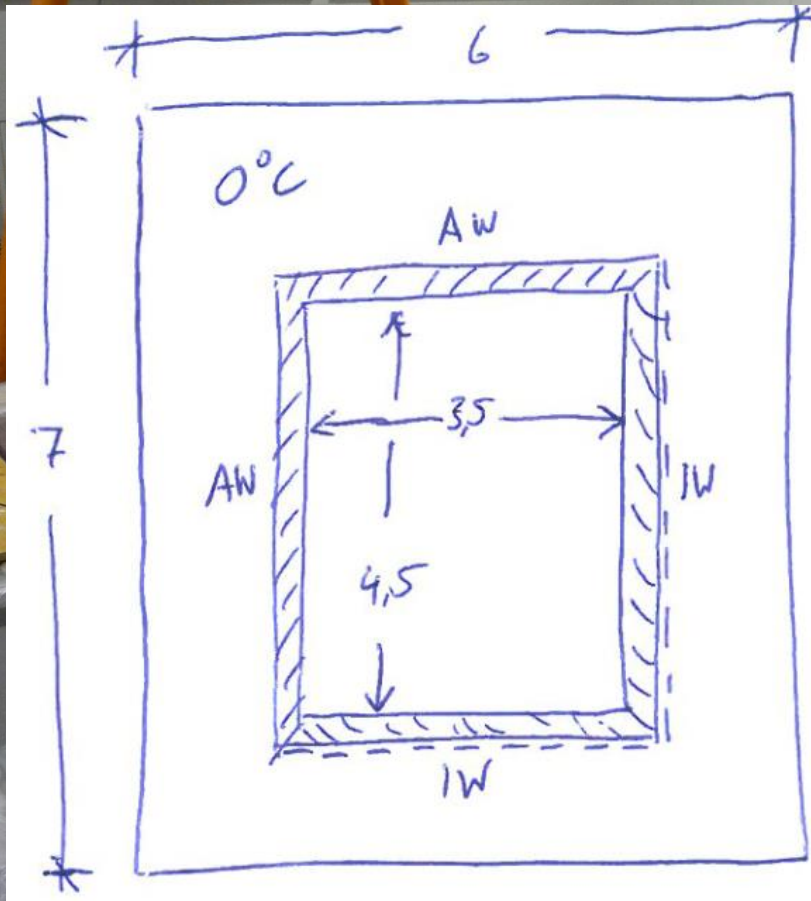
- Variety of different test cases
- Comparison values for
  - heating/cooling load
  - temperature

# Validation and Application Experiment

- Impact of interior thermal insulation and intermittent heating on
  - Interior climate conditions
  - Energy savings
  - Comfort
- Varying interior insulation systems and types
- Comparison of heating systems
- Influence of interior surface coatings
  
- Methods:
  - Measurements
  - Building and component simulation



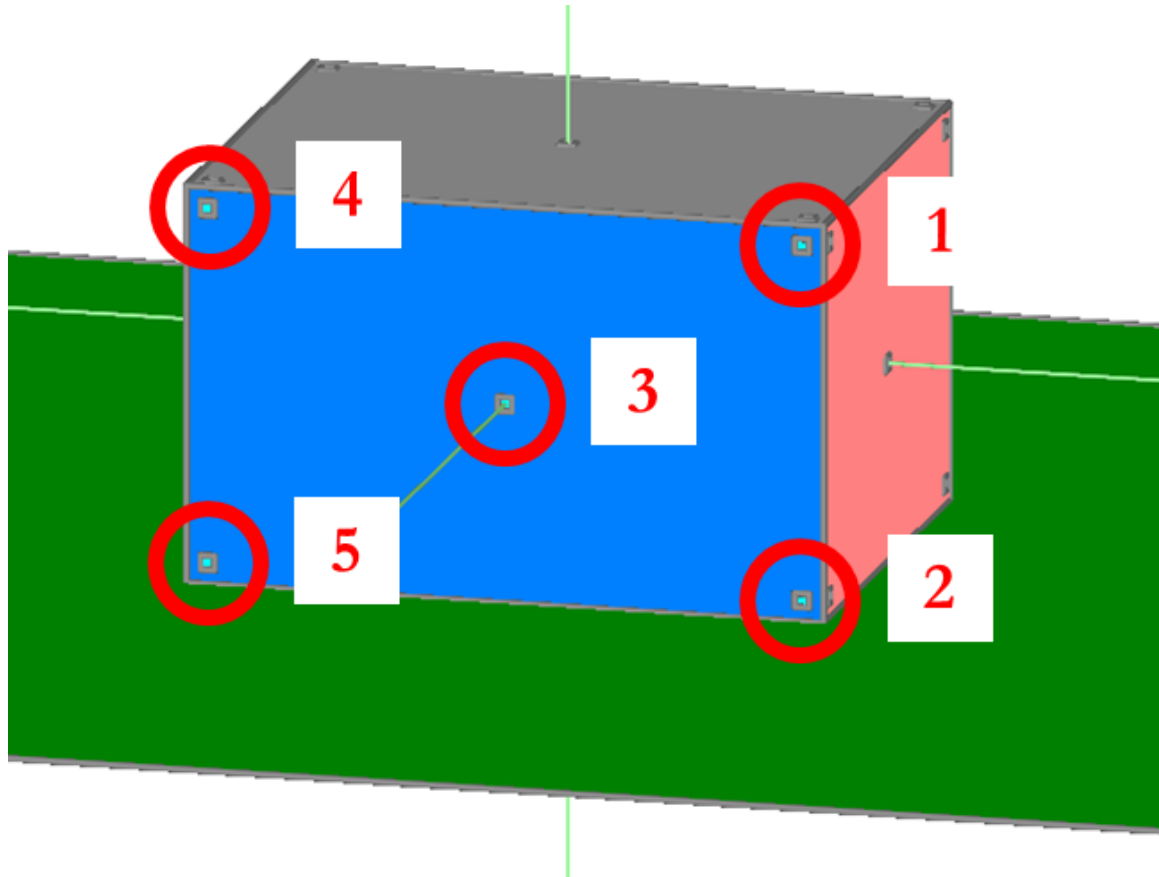
# Validation and Application Experiment



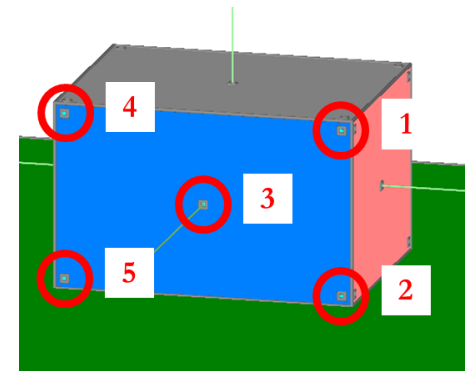
- Room with 3,5 x 4,5 m in big climate chamber
- Chamber (outside) temperature
- Inner walls (on the outside) temperature 20 °C

- approx. 160 temperature sensors
- 25 convective heat transfer sensors
- 10 controls for heating surfaces
- Humidity sensors
- Heat flux transducers

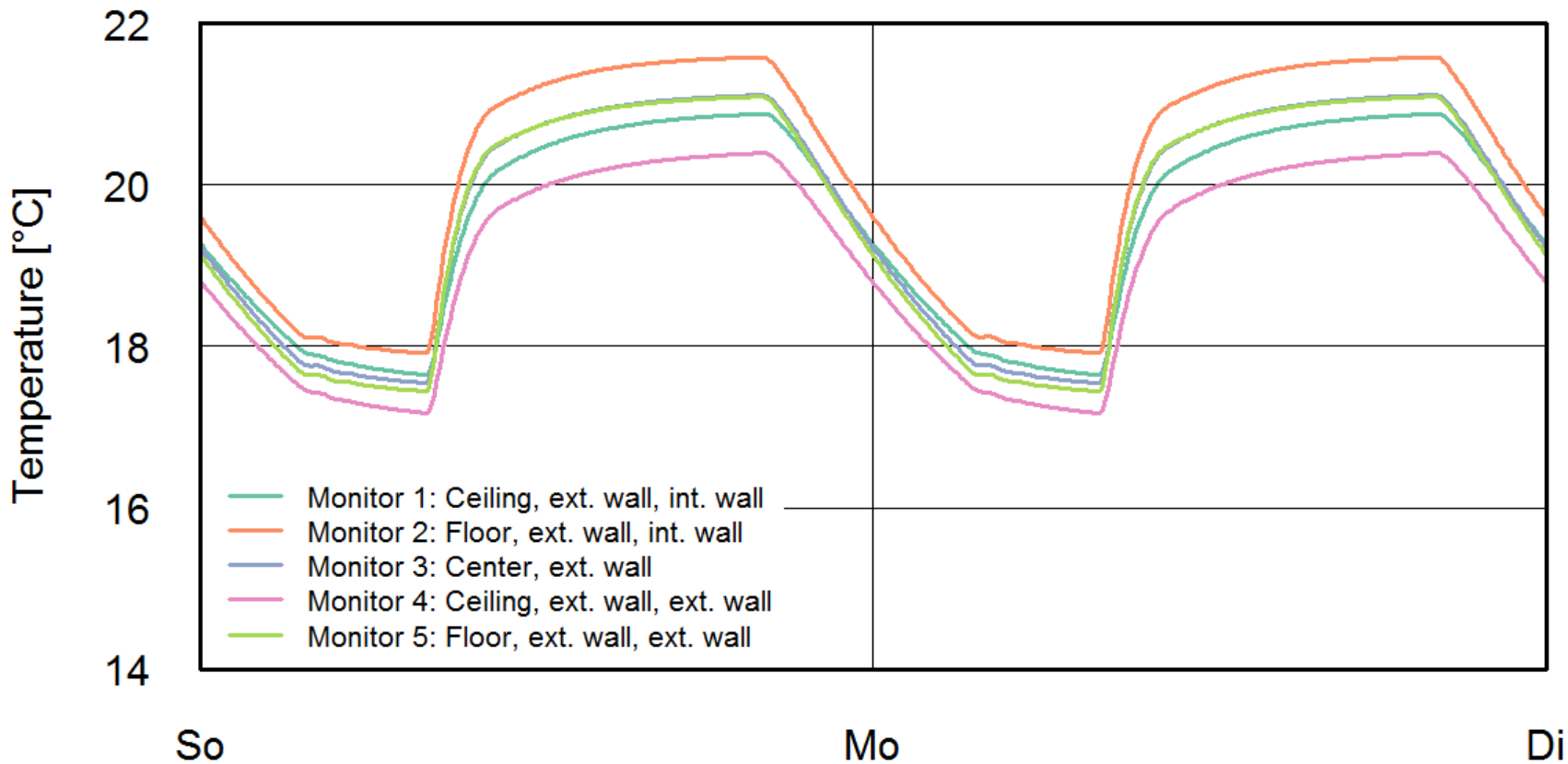
# Validation and Application Experiment



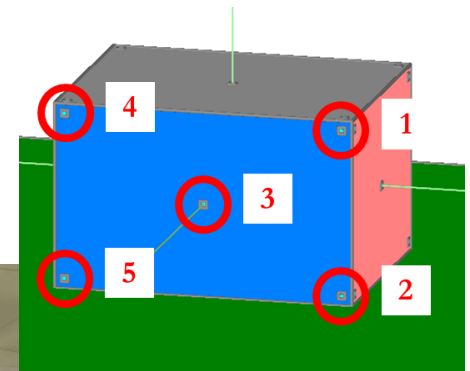
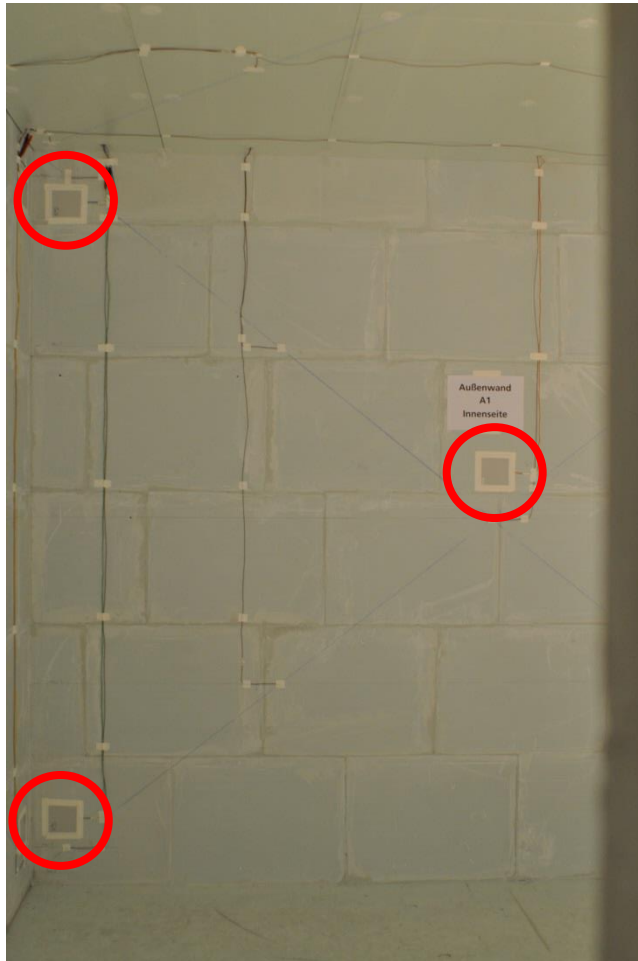
# Validation Experiment – Expected Range



Interior surface temperature



# Experiment Instrumentation



# Experiment Instrumentation



# Application Experiments - Goals

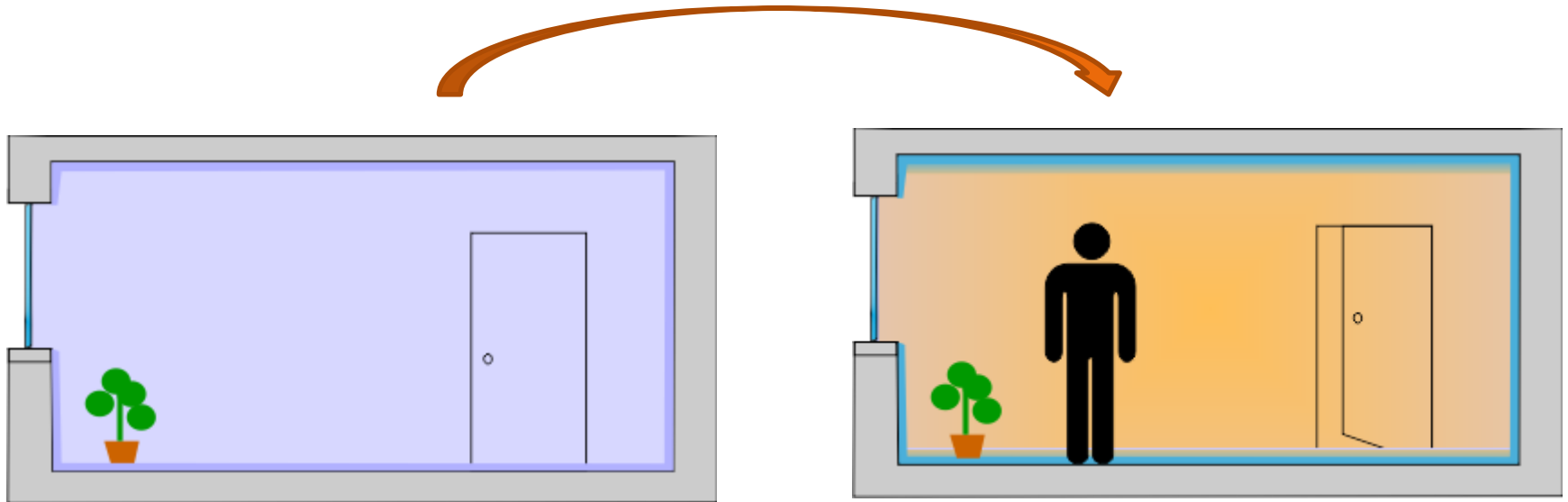
Development of **efficient strategies** for **intermittent building use**, e.g.:

- Quick reacting panel heating systems
- Insulation of massive components (also interior)
- Coatings for inner surfaces (e.g. low-e)

→ Assess options with **experiment** und **simulation**

→ Take **restraints** (e.g. humidity, comfort) into account

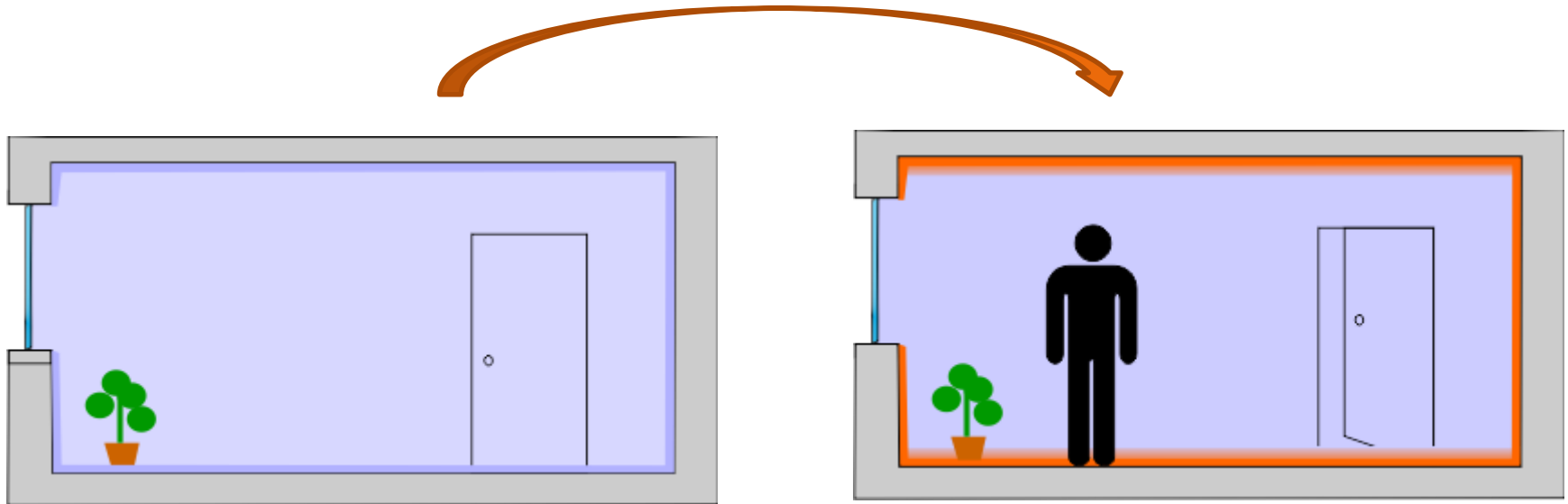
# Current approach



- Unconditioned space
- Cold/Warm Air and Surfaces

- Blow in cold/warm air to condition the space
  - Surface conditions change slower
  - Air temperature needs to fix it

# Intermittent Heating/Cooling of a Space

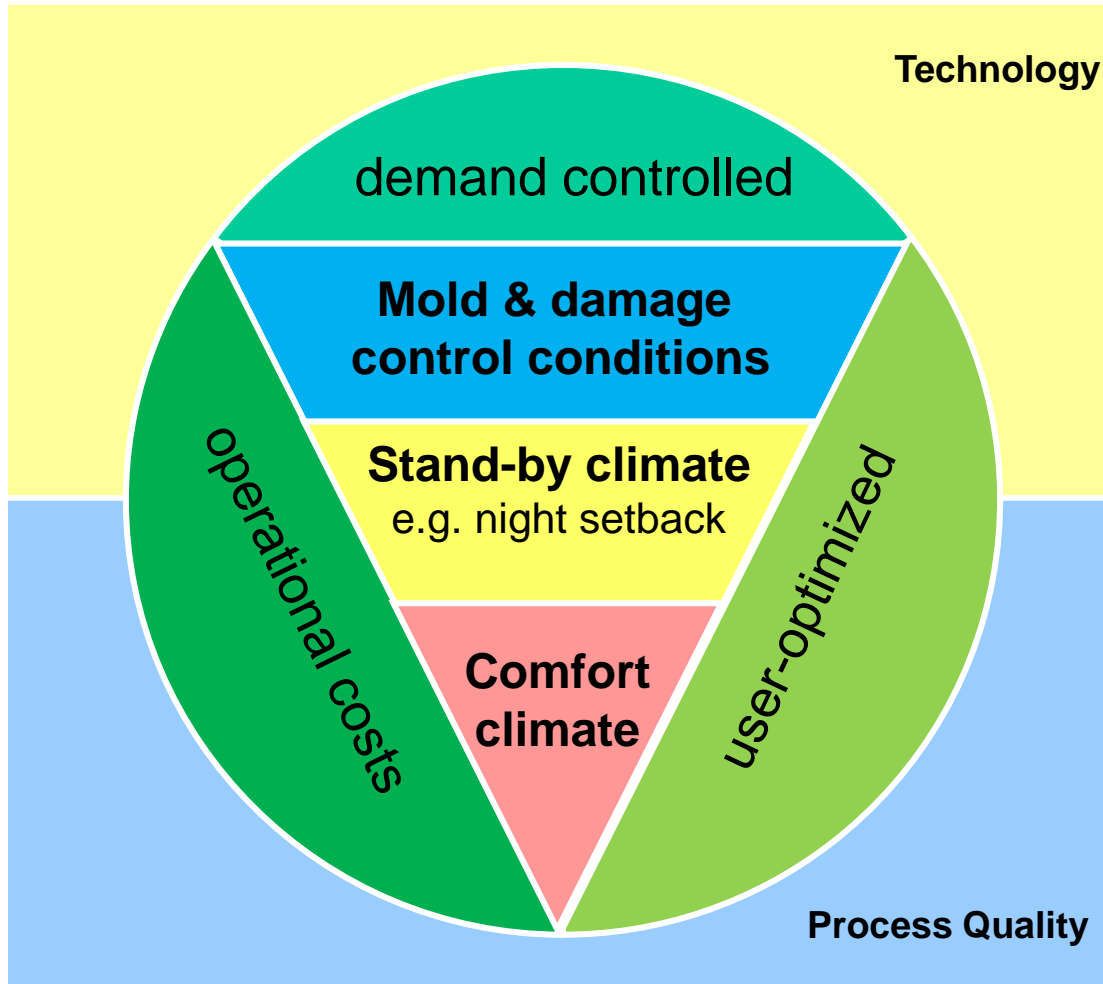


- Unconditioned space
- Cold/Warm Air and Surfaces

- Condition the surfaces
  - Low-e surface treatment
  - Thin internal insulation of all surfaces
  - Panel heating/cooling on exterior wall surface



# Where do we want to get?



# Summary and Conclusion

- Model for interior longwave heat exchange implemented in WUFI® Plus
  
- Model successfully validated:
  - Comparative calculations with analytical solutions
  - Validation tests from international standards (DIN, ASHRAE)
  
- Measurements will be used for further validations
  
- Validated model will be used for development of new and innovative demand responding space conditioning strategies

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# Implementation and validation of a long-wave heat exchange model

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## Contact:

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Building on knowledge

